

# Wood Stoves for Future Energy Efficient Buildings

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Highly efficient and clean wood log stoves, Expert Workshop  
IEA Bioenergy Task 32, 29 October 2015, Berlin, Germany

# Towards highly-insulated buildings

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- Building concepts and regulation:
  - Currently the TEK10 building regulation is into force
  - The *Norwegian Passive House* (NS3700) standard seen as a basis for the next **TEK15** building regulation (under development)
  - In 2020, all new buildings should be *Nearly-Zero Energy* (nZEB): “A building with very high energy-efficiency where the reduced energy use is significantly covered by renewable energy sources onsite or nearby”.
  - R&D on zero emission buildings (ZEB) within the Norwegian ZEB FME center

In Nordic countries, we can conclude that future buildings envelopes will be highly-insulated



The Research Centre on  
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# Highly-insulated building envelope

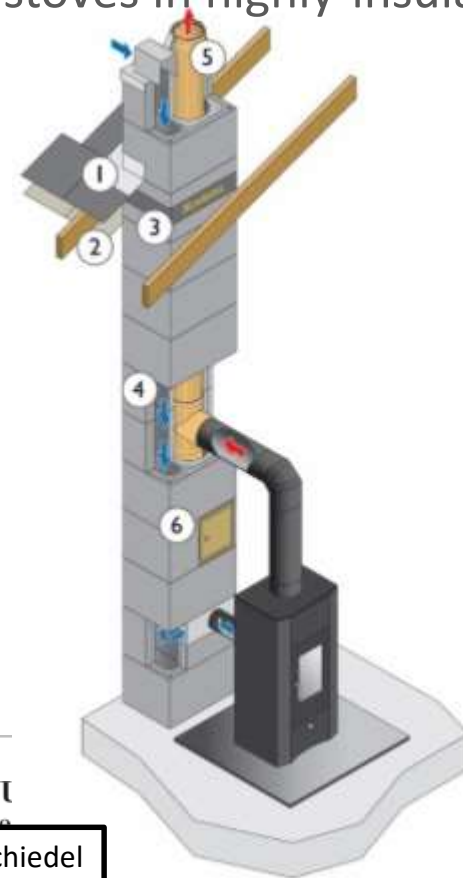
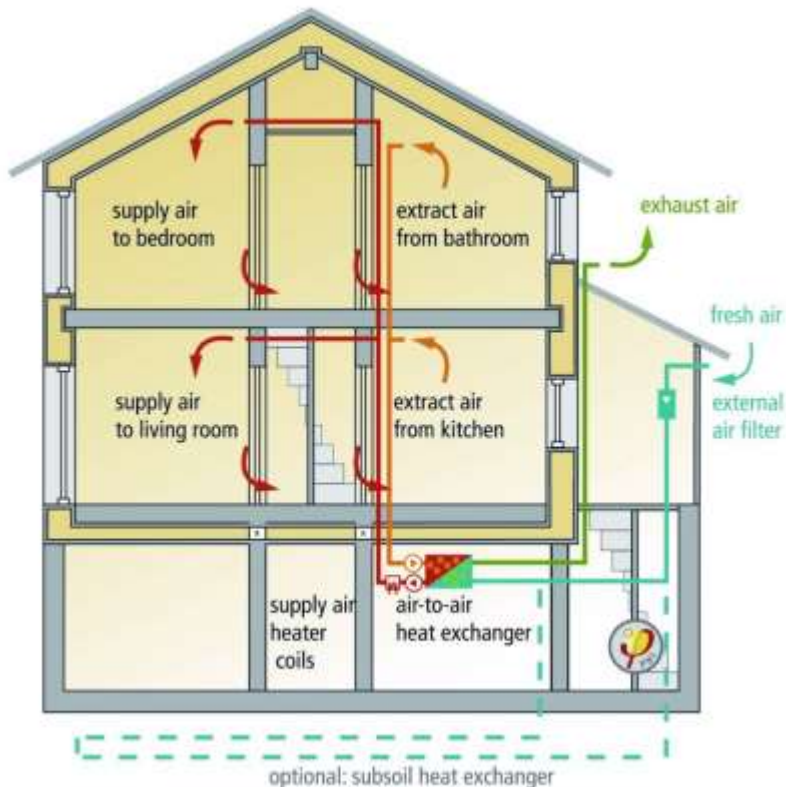
- Example of the Norwegian Passive House standard
  - Only requirements on the building envelope
  - Nothing is said about the energy efficiency of the space-heating system

Minimal requirements	TEK 10 (14.3)	NS 3700:2013
U-value global	-	$\leq 0.48 \text{ W/m}^2_{\text{BRA}} \cdot \text{K}$
U-value walls	$\leq 0.18 \text{ W/m}^2 \cdot \text{K}$	0.10-0.12*
U-value floor	$\leq 0.15 \text{ W/m}^2 \cdot \text{K}$	0.08-0.09*
U-value roof	$\leq 0.13 \text{ W/m}^2 \cdot \text{K}$	0.08*
U-value windows (total with frame)	$\leq 1.2 \text{ W/m}^2 \cdot \text{K}$	$\leq 0.8 \text{ W/m}^2 \cdot \text{K}$
U-value doors	$\leq 1.2 \text{ W/m}^2 \cdot \text{K}$	$\leq 0.8 \text{ W/m}^2 \cdot \text{K}$
Normalized thermal bridges, $\Psi''$	$\leq 0.03 \text{ W/m}^2_{\text{BRA}} \cdot \text{K}$	$\leq 0.03 \text{ W/m}^2_{\text{BRA}} \cdot \text{K}$
Yearly efficiency heat recovery, $\eta_t$	$\geq 70\%$	$\geq 80\%$
SFP ventilation	-	$\leq 1.5 \text{ kW}/(\text{m}^3/\text{s})$
Air infiltration 50 Pa	$\leq 2.5 \text{ ach}$	$\leq 0.6 \text{ ach}$

(\*) = typical value, not a requirement

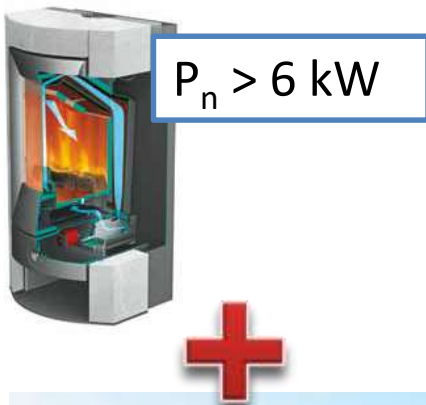
# Stove integration in highly-insulated houses (1)

- Challenge for **Indoor Air Quality (IAQ)**:
  - Airtight building envelopes equipped with **balanced mechanical ventilation**
  - The stove envelope should be airtight
  - Independent air circuits for combustion air and flue gas removal
  - Lack of measurements on IAQ using wood stoves in highly-insulated buildings



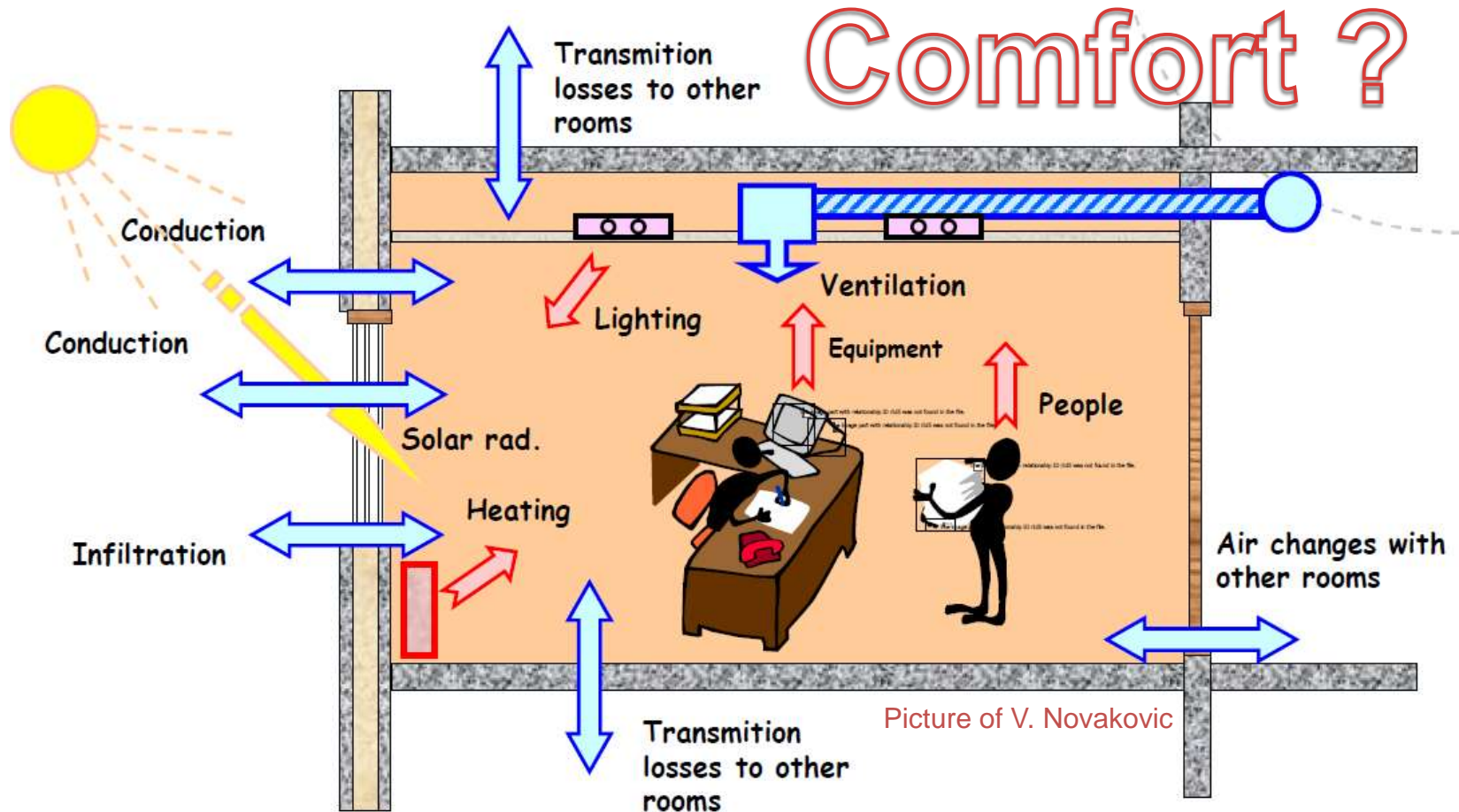
# Stove integration in highly-insulated houses (2)

- Challenge for **Indoor Thermal Environment**:
  - Nominal Power ( $P_n$ ) is oversized for passive houses
    - Ex. 6-8 kW for pellet stoves and 4-8 kW for wood-log stoves
    - Ex. 160 m<sup>2</sup> detached passive house has ~3kW space-heating power for Oslo climate in design weather conditions (i.e. cold wave)
  - Power modulation is limited and should operate on long combustion cycles
  - Risk of overheating in the room where the stove is placed
  - But potential to simplify space-heating distribution using wood stoves



# Buildings are complex multi-physical systems

- Thermal dynamics of the building envelope



# Thermal comfort assessment

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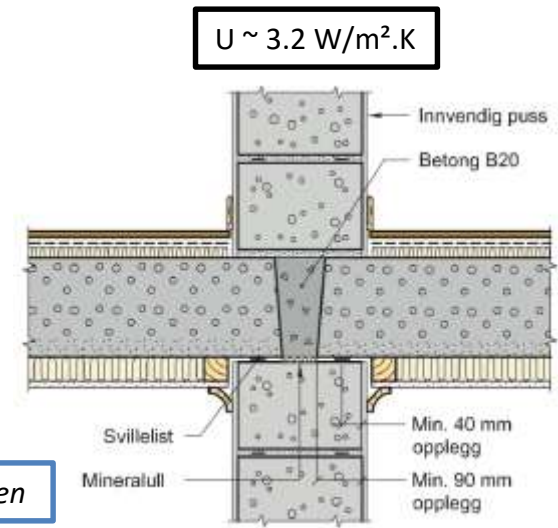
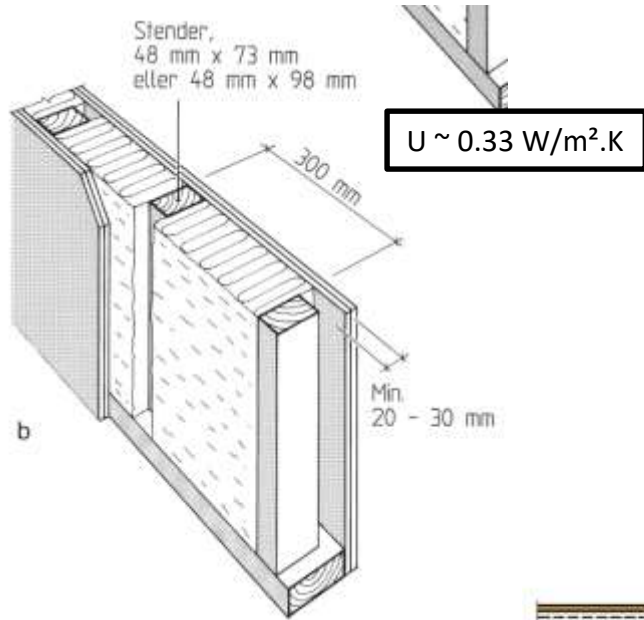
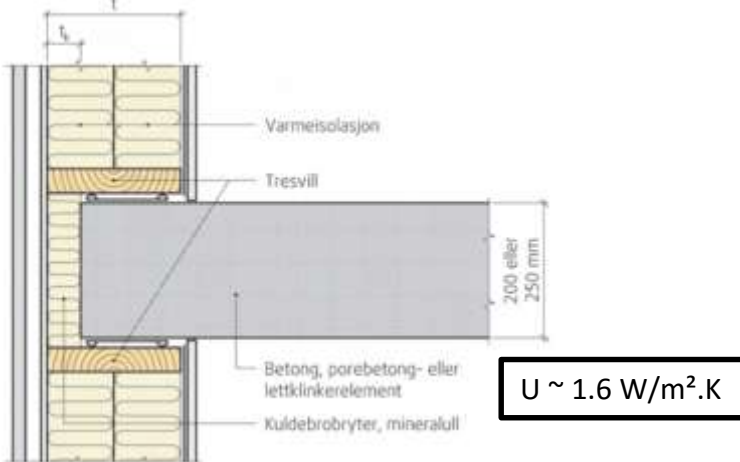
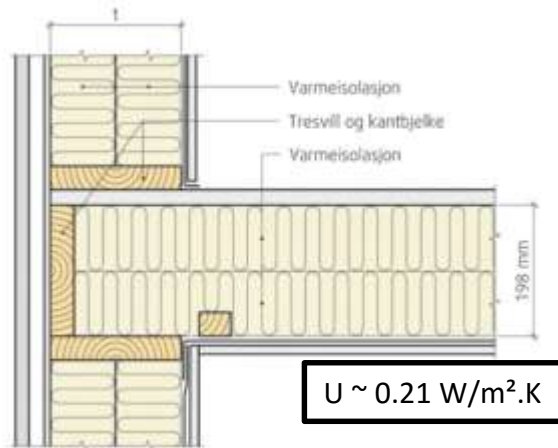
- Usually based on Fanger's approach (ISO 7730)
- **Global** thermal comfort
  - **Operative temperature (main index here)**
  - PMV and PPD (with Clo and MET)
  - Typical thermal comfort index in building performance simulation

$$T_{op} = (h_c T_{air} + h_r T_{mrt}) / (h_c + h_r) \approx 0.5 (T_{air} + T_{mrt})$$

- **Local** thermal comfort
  - Based on wall temperatures
    - Radiant asymmetry
    - Cold/warm floor
  - Based on air and temperature fields
    - Draft
    - Vertical temperature stratification

# Heat conduction through internal walls (1)

- Construction of partition walls and ceilings
  - With wooden walls, insulation against noise propagation (mineral wool)



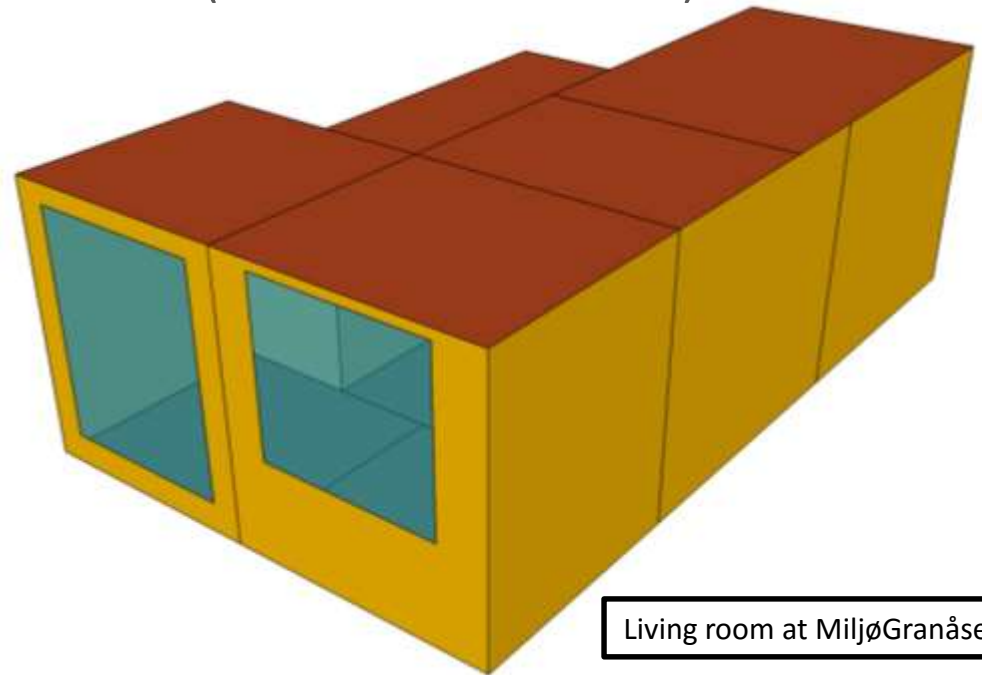
Pictures from Byggforskserien



# Heat conduction through internal walls (2)

- Example

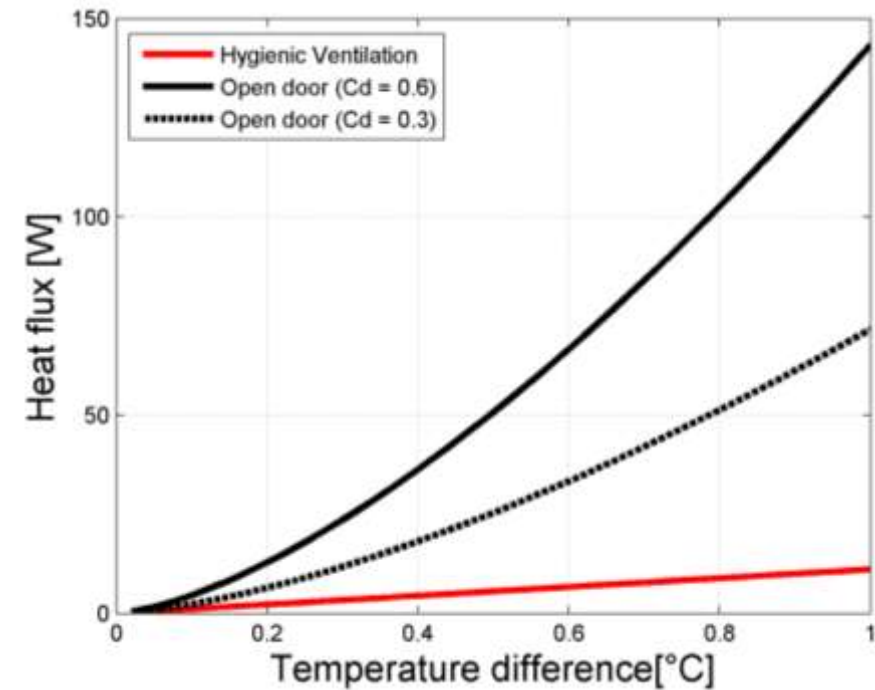
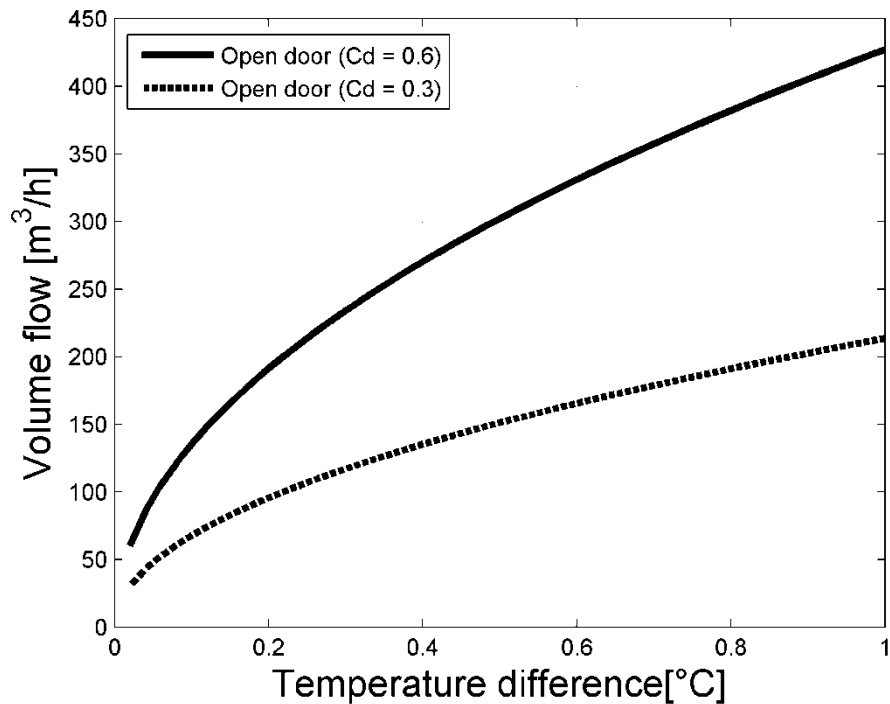
- Partition ceiling of 32.5 m<sup>2</sup>
- Partition wall of 24.14 m<sup>2</sup>
- Concrete construction :  $U = 32.5 * 1.6 + 24.14 * 3.2 = 129 \text{ W/K}$
- Wood construction:  $U = 32.5 * 0.21 + 24.14 * 0.33 = 15 \text{ W/K}$
- Large temperature differences needed (for > 1 kW to transfer)
- Risk of overheating



Living room at MiljøGranåsen

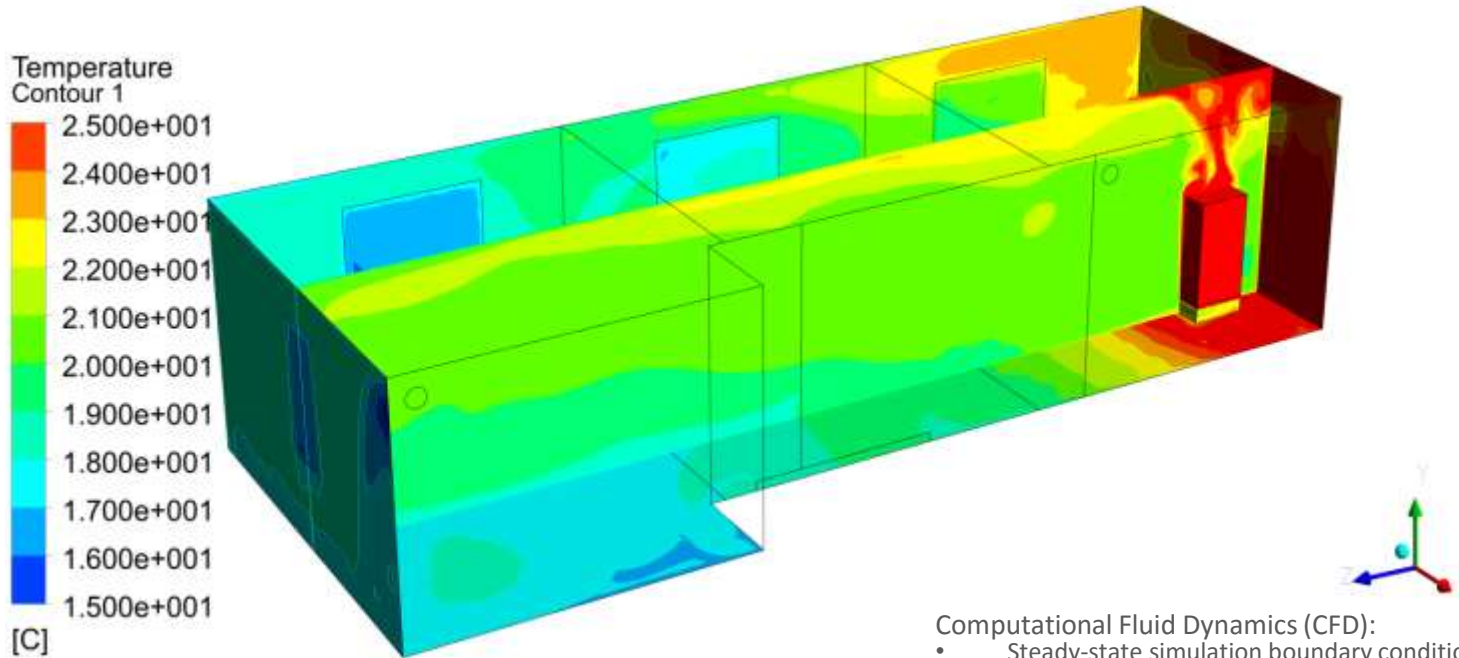
# Heat flow through open doors (2)

- Mass flow and convected energy
  - Door (0.9 x 2.35) m<sup>2</sup>
  - More power can be transferred but still not enough (for  $P > 1\text{kW}$ )



# Temperature distribution inside room (1)

- Strong gravity currents: assuming no obstacle on the ceiling
  - The temperature difference in the horizontal direction is small
  - The temperature field is essentially 1D (in vertical direction)



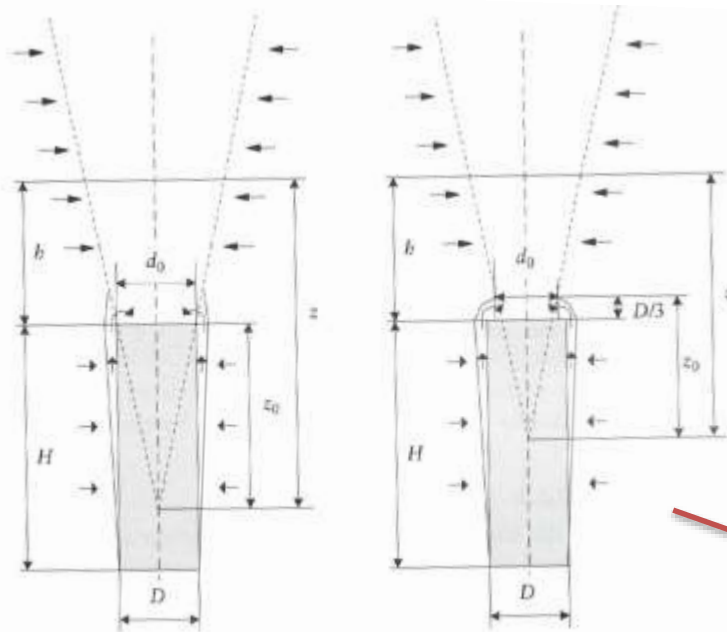
Snapshot of the unsteady temperature field

Computational Fluid Dynamics (CFD):

- Steady-state simulation boundary conditions (stove not oversized)
- Boundary conditions pre-computed using TDS (decoupled)
- Unsteady flow due to high Rayleigh number
- URANS using the k- $\epsilon$  RNG model on a  $1.0 \cdot 10^6$  nodes tetrahedral mesh
- *Georges et al., BSO 2012 conference, Loughborough*

# Temperature distribution inside room (2)?

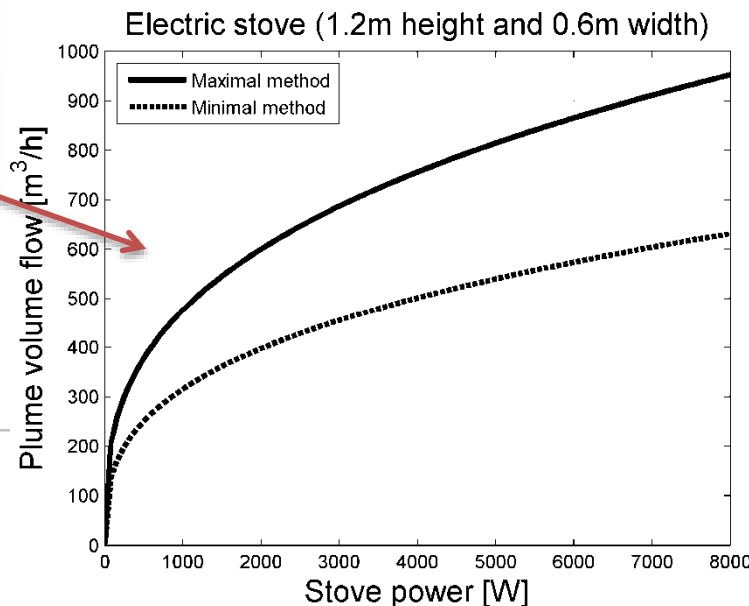
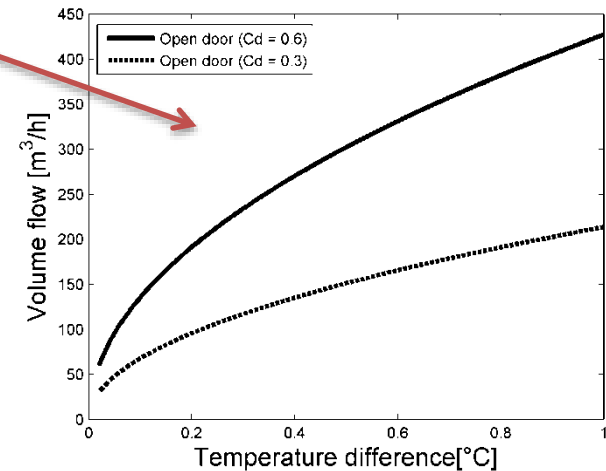
- Mass flow rates: flow controlled by the stove thermal plume
  - Hygienic ventilation (TEK 10):  $(1.2\text{m}^3/\text{h}\cdot\text{m}^2) \cdot 32.5\text{m}^2 = 39\text{ m}^3/\text{h}$
  - Bidirectional flow open door: 200-600  $\text{m}^3/\text{h}$
  - Thermal plume of the stove: 500-1000  $\text{m}^3/\text{h}$



(a) Maximum case

(b) Minimum case

FIGURE 7.65 Convection flow above a vertical cylinder.



Industrial Ventilation: Design Guidebook (2001)



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# How to prevent overheating?

- Passively, the heat cannot be sufficiently transported
  - Through partition walls and ceilings
  - Through open internal doors

- Overheating can should avoided by
  1. Storing heat in the building walls (passive approach)
  2. Storing heat in the stove walls (cf. work Øyvind)
  3. Storing in water using a hydro-stove (active approach)
  4. Flatten the heat release from combustion

- (1) Storing heat in walls
  - Comfort based on **operative temperature** ( $T_{op}$ )
  - Thermal mass to reduce wall temperature
  - Slow down the temperature increase of air
  - Transients should be analyzed by *thermal dynamic simulation* (i.e. no easy analytical solution)

$$T_{op} \square \frac{T_{sens}^{air} + T_{mrt}^{wall}}{2}$$

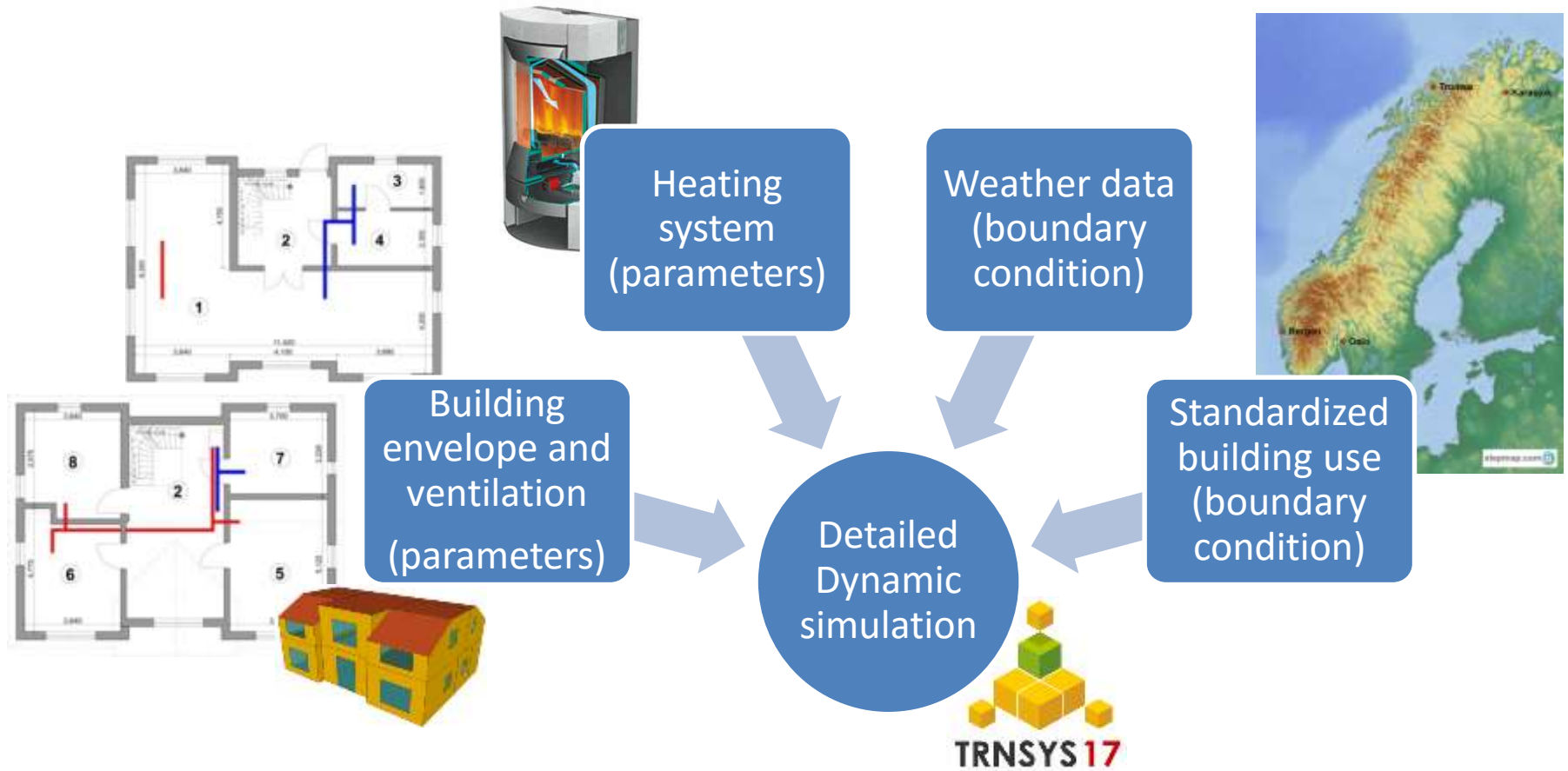
# Modeling approaches

- Advantages and limitations of the main modeling approaches
  1. Thermal dynamic simulation (TDS)
  2. TDS + zonal model in rooms
  3. CFD
  4. CFD + TDS

Method	$\Delta t$ imposed by	Tmin	Tmax	CPU time	Convection doors	Consistent BCS	Top	Stratification	Radiation asymmetry	Air velocity
TDS	Control/Flow	1-cycle	1-year	Low-Medium	Simple	Yes	Yes	No	Yes	No
Zonal TDS	Control/Flow	1-cycle	1-year	Low-Medium	Simple	Yes	Yes	<i>Maybe</i>	Yes	<i>Maybe</i>
CFD	Flow	1-cycle	Few cycles	High	Accurate	No	Yes	Yes	Yes	Yes
TDS+CFD	Flow	1-cycle	Few cycles	High	Accurate	Yes	Yes	Yes	Yes	Yes

Tmin = minimal simulation time; Tmax = maximal simulation time

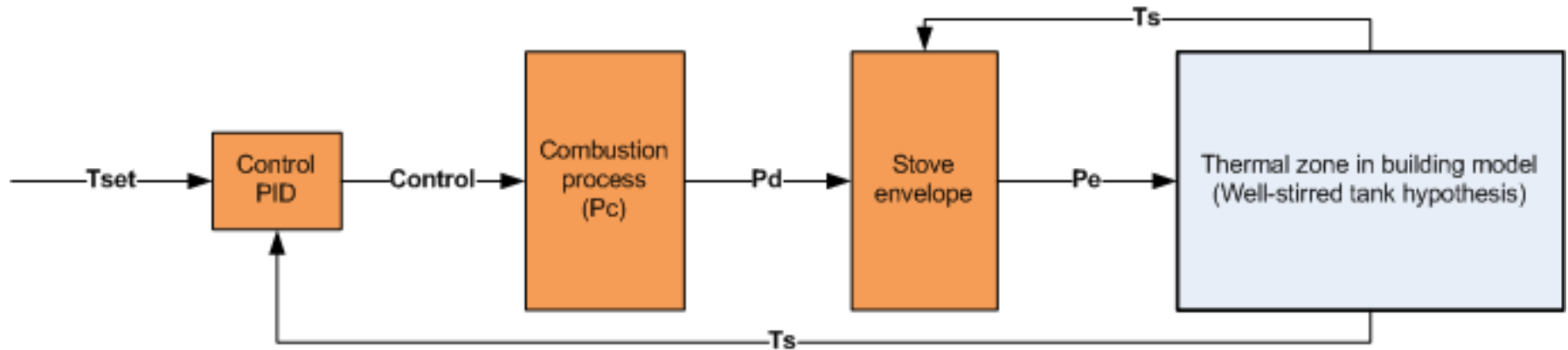
# Thermal dynamic simulations (1)



- Coupled simulation of the building envelope, the ventilation and the stove during the entire heating period using TRNSYS

# Thermal dynamic simulation (2)

- Insight into all-year thermal comfort at acceptable CPU cost

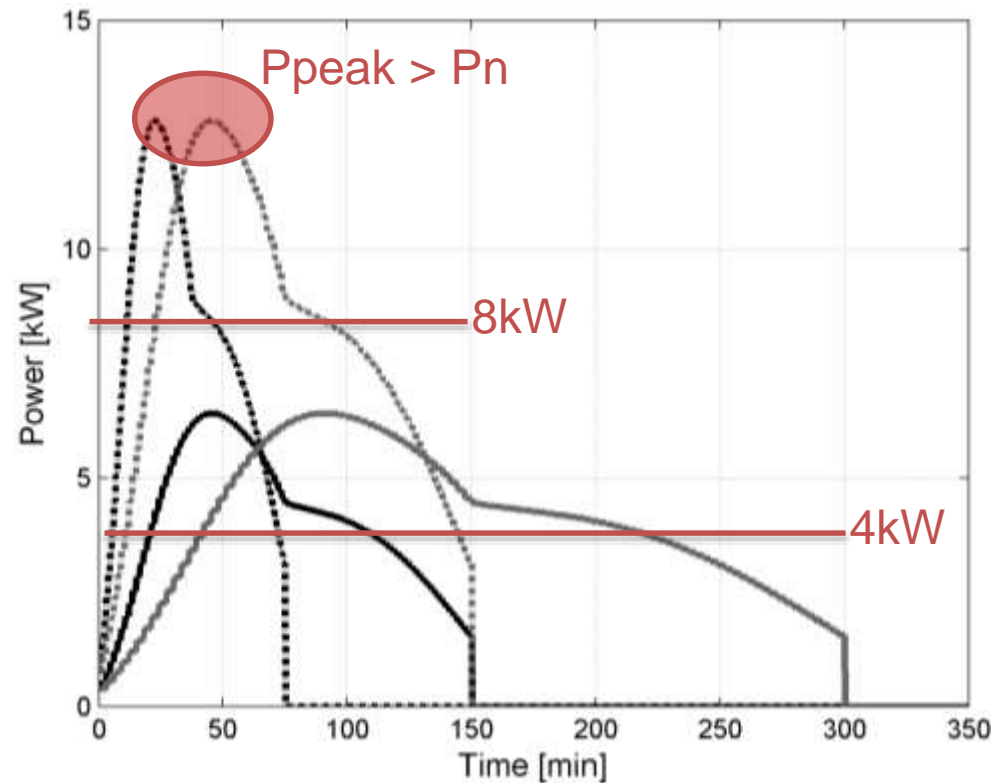


- Type56b for the building model
- PID control of power modulation for a pellet stove, manual for log stoves
- Batch combustion model for wood logs
- 1-D heat transfer in the stove envelope
- Correlation for convection from the stove surface to the room
- Detailed view factors evaluation from the stove to room surfaces and user



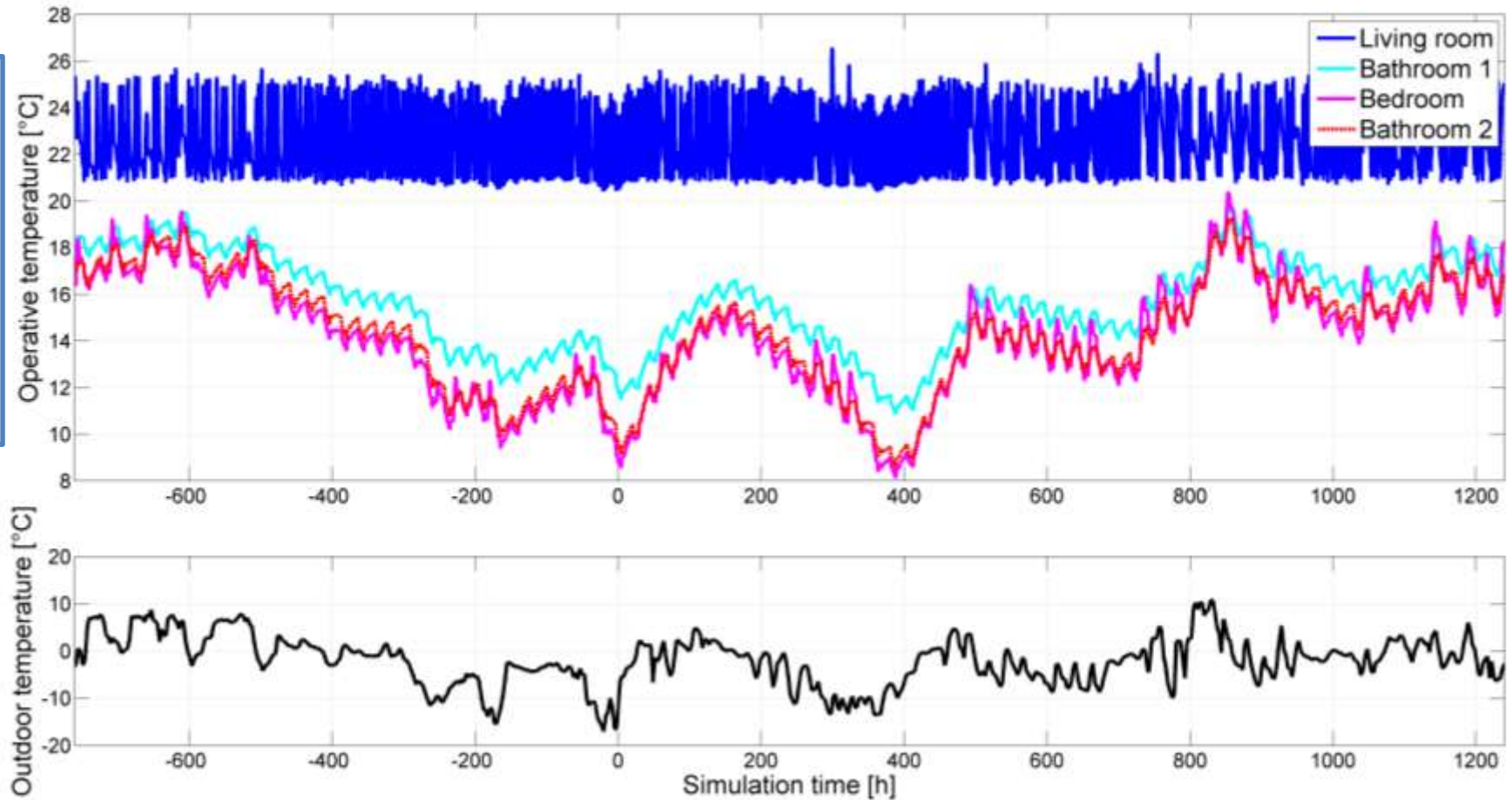
# Thermal dynamic simulation (3)

- Batch combustion model for wood logs
  - Developed by Øyvind Skreiberg (SINTEF Energy Research)
  - Semi-empirical model with different phases:
    - Drying
    - Pyrolysis/devolatilization
    - Char oxidation/gasification
  - Example of the 8 kW of  $P_n$ 
    - No modulation (dashed)
    - 50% modulation (solid)
    - 20 kWh batch load (grey)
    - 10 kWh batch load (black)



# Thermal dynamic simulations (4)

- Example result:
  - For the climate of Oslo, only heated by stove
  - Stove of 8kW with 30% power modulation
  - Wooden construction, closed internal doors



# Thermal dynamic simulations (5)

- Matrix risk overheating
  - Location
  - Stove thermal properties
  - Building architectonic properties
  - Period of the year



On the proper integration of wood stoves in passive houses under cold climates

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**Table 7**

Qualitative performance against overheating of log stoves equipped with 50 kJ/K thermal inertia: function of the stove nominal power,  $P_{d,n}$  in [kW], power modulation [%] and batch load,  $Q_d$  in [kWh], computed for different locations and weather conditions.

Location Weather	Oslo			Bergen			Karasjok		
	TMY	CTMY	SDC	TMY	CTMY	SDC	TMY	CTMY	SDC
4 kW, 50%, 10 kWh	⊙	⊙	⊙	⊙	⊙	⊙	⊙	⊙	⊗
4 kW, 100%, 10 kWh	⊙⊙	⊙⊙	⊙⊙	⊙⊙	⊙⊙	⊙⊙	⊙⊙	⊙⊙	⊙
4 kW, 50%, 5 kWh	⊙	⊙	⊕	⊙	⊙	⊕	⊙	⊙	⊕
4 kW, 100%, 5 kWh	⊙	⊙	⊙	⊙	⊙	⊙	⊙	⊙	⊕
8 kW, 50%, 20 kWh	⊙⊙	⊙⊙	⊙⊙	⊙⊙	⊙⊙	⊙⊙	⊙⊙	⊙⊙	⊙⊙
8 kW, 100%, 20 kWh	⊖	⊖	⊖	⊖	⊖	⊖	⊖	⊖	⊖
8 kW, 50%, 10 kWh	⊙⊙	⊙⊙	⊙⊙	⊙⊙	⊙⊙	⊙⊙	⊙⊙	⊙⊙	⊙⊙
8 kW, 100%, 10 kWh	⊖	⊖	⊖	⊖	⊖	⊖	⊖	⊖	⊖
12 kW, 50%, 30 kWh	⊖	⊖	⊖	⊖	⊖	⊖	⊖	⊖	⊖
12 kW, 100%, 30 kWh	⊖	⊖	⊖	⊖	⊖	⊖	⊖	⊖	⊖
12 kW, 50%, 15 kWh	⊖	⊖	⊖	⊖	⊖	⊖	⊖	⊖	⊖
12 kW, 100%, 15 kWh	⊖	⊖	⊖	⊖	⊖	⊖	⊖	⊖	⊖

⊕: good control independent of the architectural measures.

⊙: control if one set of architectural measures is taken, ⊙⊙: control if complete set of architectural measures taken.

⊖: overheating whatever the architectural measures taken.

⊗: underheating because lack of emitted power.

# Validation

- **Measurements:**
  - Miljø Granåsen project in Trondheim
  - Building of 142 m<sup>2</sup> heated area
  - Measurements Mars-April 2013
  - Lightweight wooden structure
  - Unoccupied without furniture



Basement



Ground floor



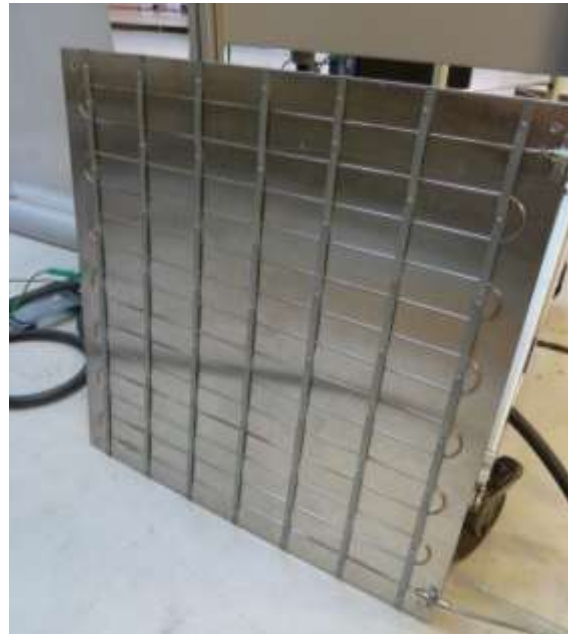
First floor

# Movable electric stove (1)

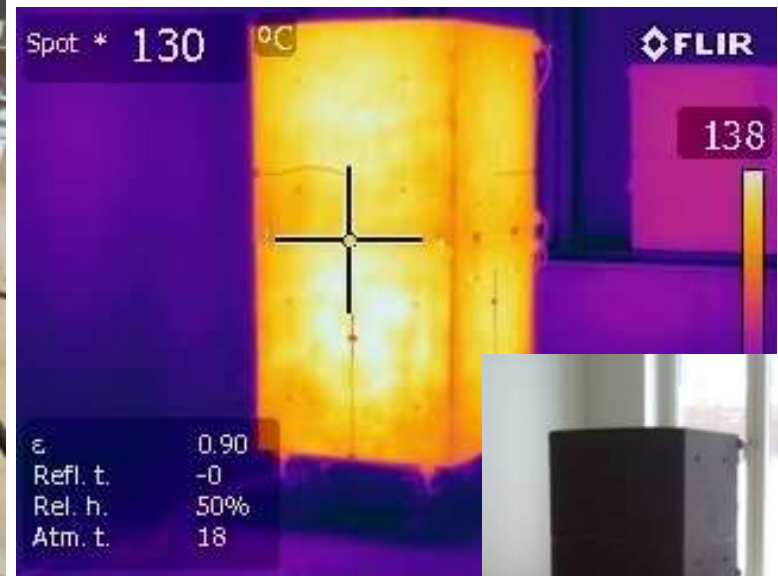
- Electric stove heater, advantages:
  - Does not need to be connected to a stack/chimney
  - Can be implemented temporarily and applied different heat release profiles
  - Electricity enables to control the heat release profile accurately
  - No risk for the IAQ



15.5 kWe and low thermal mass for quick reaction time

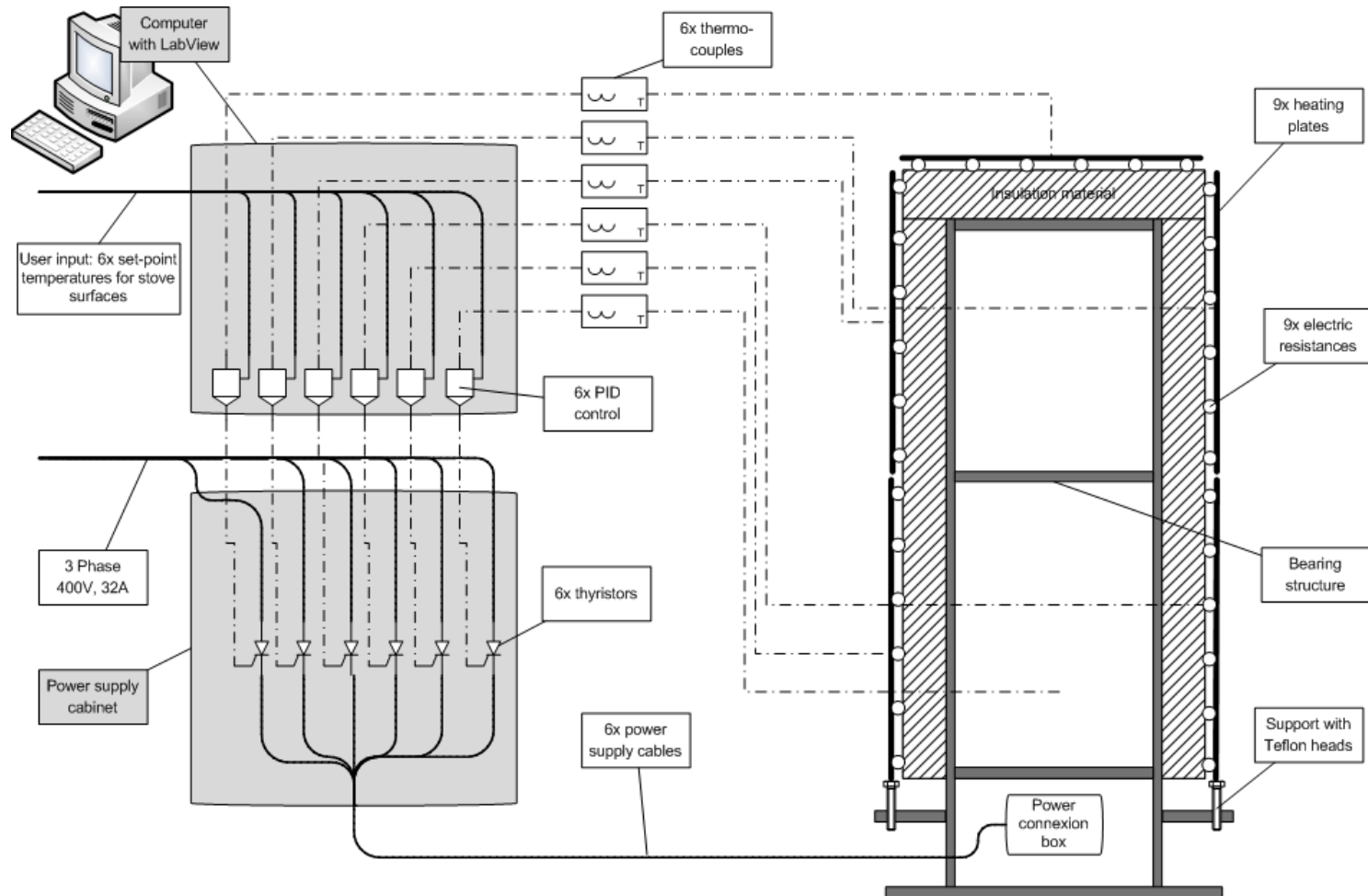


Good temperature distribution on stove surfaces



# Movable electric stove (2)

- Principle:



# Movable electric stove (3)



- Measurements in passive house
  - Air temperature distribution in room
  - Wall temperature distribution in room
  - Flow through open door between ground and first floor

Type	Number	Location	Precision	Measure
PT-100	5	Ground floor	$\pm 0.1^{\circ}\text{C}$	Ts, stratification
	5	Staircase	$\pm 0.1^{\circ}\text{C}$	Ts, stratification
	1	Living-room	$\pm 0.1^{\circ}\text{C}$	Ts, 0.8 m height
	1	Kitchen	$\pm 0.1^{\circ}\text{C}$	Ts, 0.8 m height
	1	Kitchen	$\pm 0.1^{\circ}\text{C}$	Top, 0.8 m height
	7	Walls	$\pm 1^{\circ}\text{C}$	Twall
Radiant temperature transducer INNOVA MM0036	1	Living-room and kitchen	$\pm 0.5^{\circ}\text{C}$	Tmrt, 0.8 m height
Thermocouples Type T	10	Doorway or living-room	$\pm 1\%$ $\pm 0.5^{\circ}\text{C}$	Ts, profile or stratification
Anemometer TSI 8475	10	Doorway	$\pm 3\%$ $\pm 0.005 \text{ m/s}$	Air velocity profile
Temperature logger iButton Maxim Integrated DS1922L	11	Each room	$\pm 0.06^{\circ}\text{C}$	Ts, one by room
	1	Outdoor	$\pm 0.06^{\circ}\text{C}$	Ts, sheltered
	3	Air Handling Unit	$\pm 0.06^{\circ}\text{C}$	Ts fresh air

# Movable electric stove (4)



- Test cases for measurements in passive house
  - Wood pellet stove (4 test cases)

Case	$P_n$	Modulation	$I_{th}$	Cycle length
N°	[kW]	[% of $P_n$ ]	[kJ/K]	[min]
1p	6	100	50	90
2p	6	100	150	90
3p	8	30	50	90
4p	8	100	150	90

- Wood log stove (8 test cases)

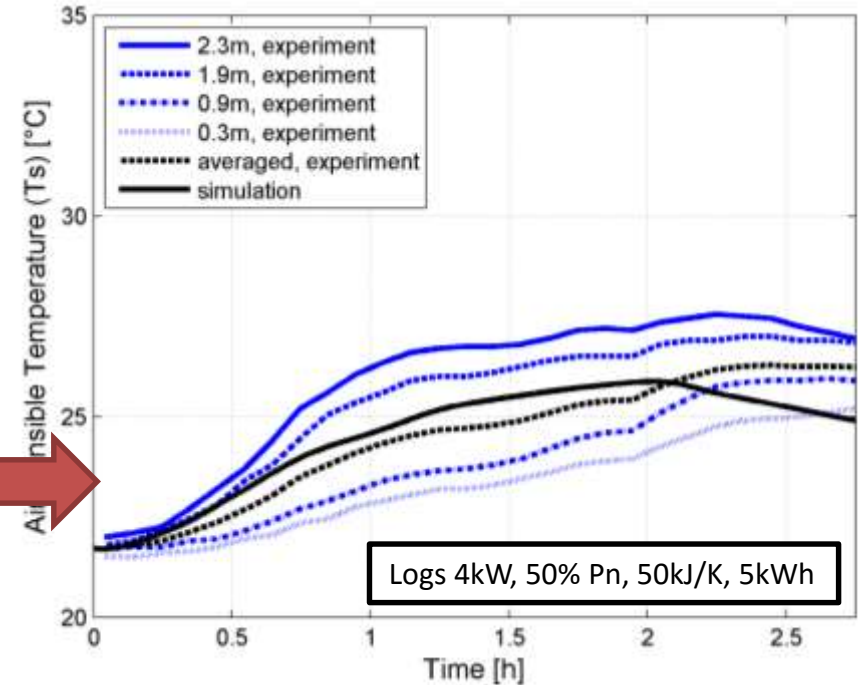
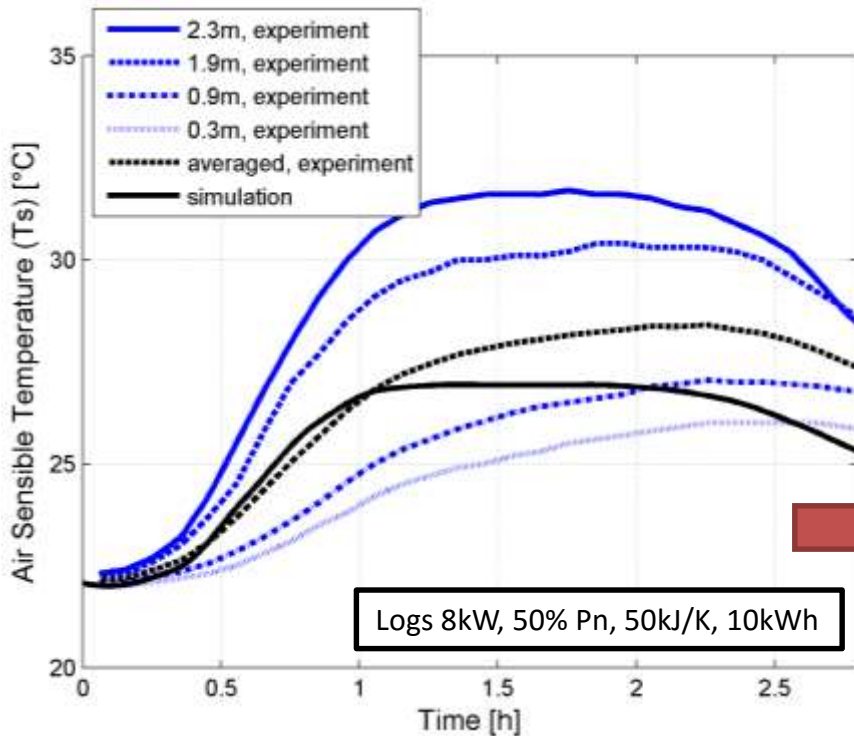
Case	$P_n$	Modulation	$I_{th}$	Batch load
N°	[kW]	[% of $P_n$ ]	[kJ/K]	[kWh]
1w	4	50	50	5
2w	4	100	50	5
3w	4	50	50	10
4w	4	100	50	10
5w	8	50	50	10
6w	8	100	50	10
7w	8	50	150	10
8w	8	100	150	10



# Movable electric stove (5)

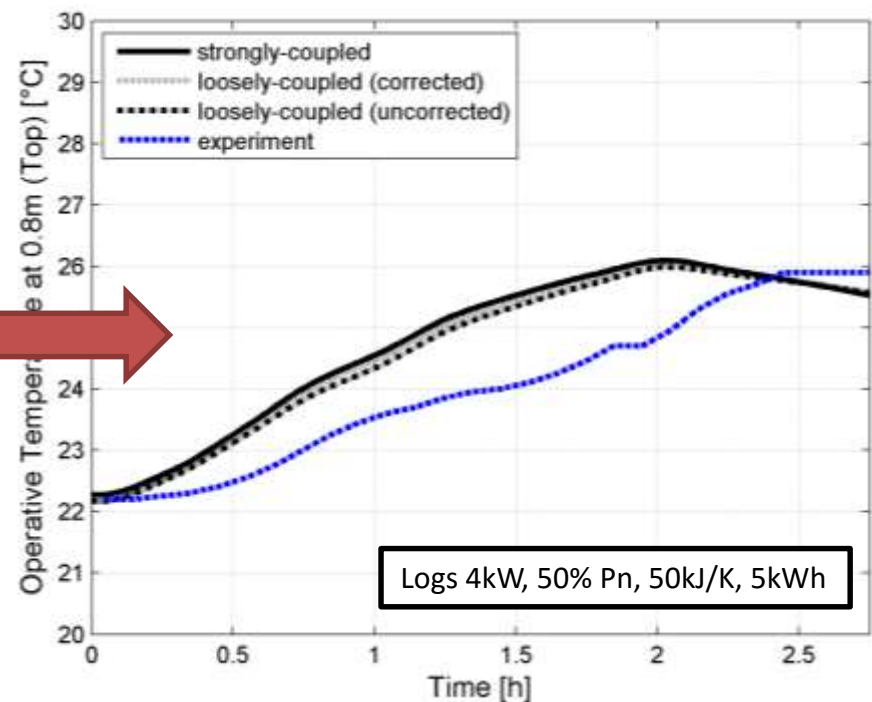
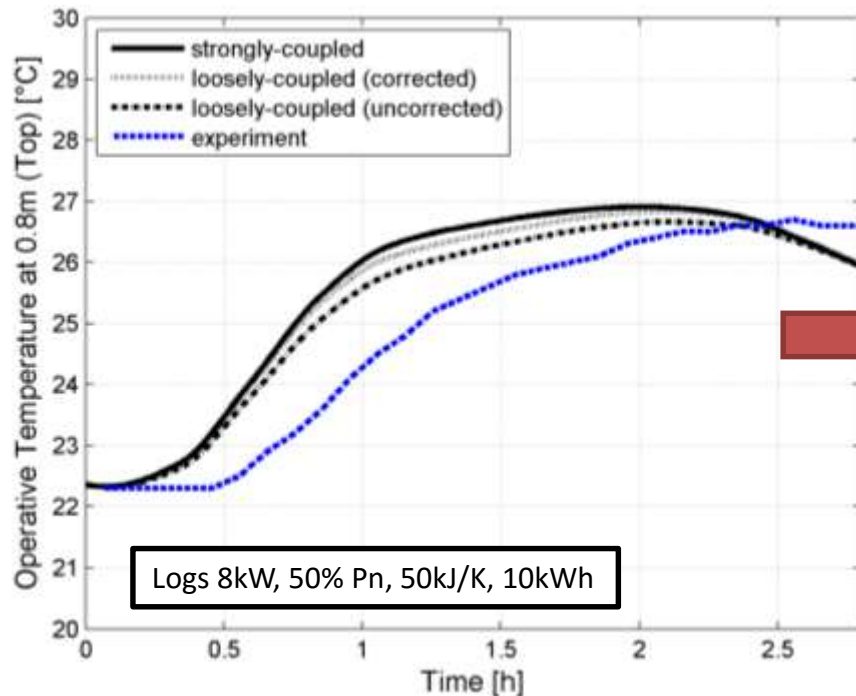


- Conclusion for temperature distribution in room
  - Significant vertical temperature gradient
  - Reduction from 8 to 4kW is efficient for wood-log stoves (with 50% Pn)
  - TRNSYS (TDS) don't capture stratification (here equal to temperature at 0.9m)



# Movable electric stove (6)

- Comparison for operative temperature in room (at 0.8m kitchen)
  - TRNSYS and measurements in good agreement in magnitude
  - Shorter time response (to be investigated)



# Movable electric stove (7)



- Conclusion for temperature distribution in room
  - Significant vertical temperature gradient
  - Small horizontal temperature gradient (except with sun)
- Vertical stratification
  - Influence the **local** thermal comfort (non negligible)
  - Enhance the heat transfer with the first floor (through ceiling)

Case	Sun	$T_{ext}$	$\Delta T_{op,max}$	$\Delta T_{s,hor,max}$	$\Delta T_{s,vert,z1,max}$	$\Delta T_{s,vert,z2,max}$
	Living room	Outside	Kitchen	Ground floor	Ground floor	Staircase
N°	[Yes-No]	[°C]	[°C]	[°C]	[°C]	[°C]
1p	No	-1	4.6	0.2	11	4.1
2p	No	+8	3.3	0.5	8.1	2.0
4p	No	+5	4.5	1.4	11	5.3
5w	No	+5	4.7	0.3	9.3	5.1
7w	No	+5	4.0	0.4	7.6	4.2
8w	No	+7	4.8	0.8	8.9	3.6
1w	Yes	+4	3.8	3.5	4.3	3.7
3w	Yes	+4	6.6	4.6	6.7	7.1
6w	Yes	+4	6.0	4.5	13	7.8

# Heat flow through open doors (1)

- Flow through large openings such as open doors
- Type of flow
  - *Bulk density flow*
  - *Boundary layer flow*

$$\text{Isothermality factor: } \frac{\Delta T}{\Delta T_w} = \frac{T_h - T_c}{T'_h - T'_c}$$

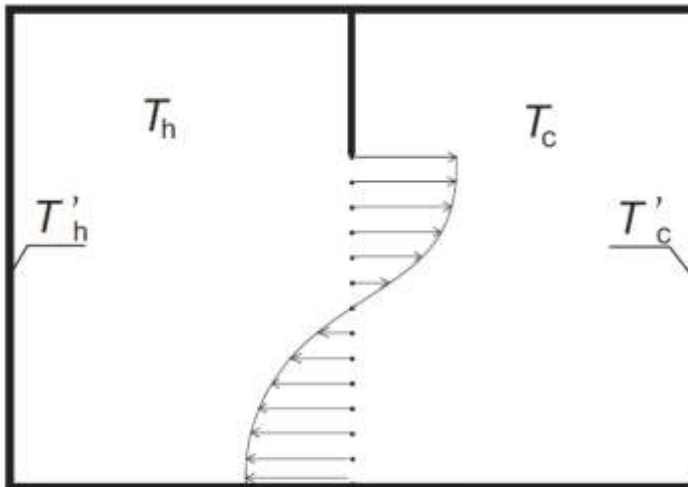


Figure 2.2 Bulk density flow  
 $\Delta T/\Delta T_w \approx 1$   
Velocity profile nearly parabolic

From PhD Clæs Blomqvist

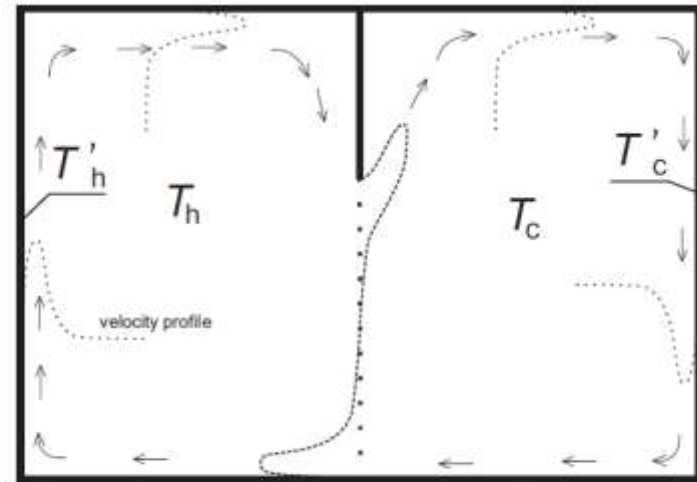
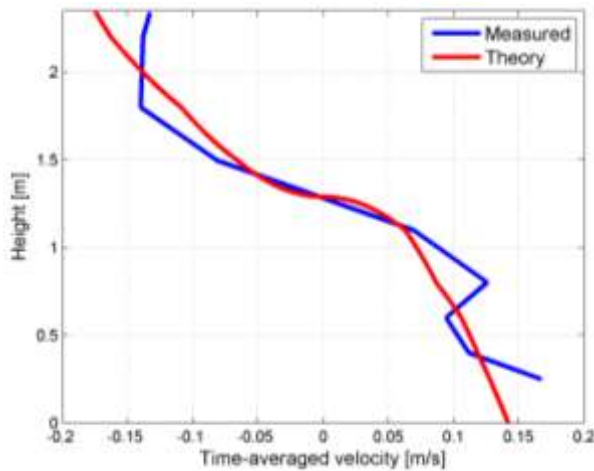


Figure 2.3 Boundary layer flow  
 $\Delta T/\Delta T_w \approx 0$   
Flow is concentrated to top and bottom of the opening

# Heat flow through open doors (3)

- Conclusion for the bidirectional airflow through open door
  - *Bulk flow and large opening approximation* correct for the mass flow
  - The convective heat exchange is underestimated if the vertical temperature stratification is not accounted for



Case	$C_{dm}$	$C_{dms}$	$C_{des}$
N°	[-]	[-]	[-]
1p	0.35	0.32	0.58
3p	0.38	0.36	0.61
1w	0.37	0.35	0.54
2w	0.38	0.33	0.62
3w	0.40	0.39	0.62
5w	0.36	0.35	0.53



# Conclusions

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- What have been done:
  1. Understand how the **heat propagates** (between rooms and inside room)
  2. Understand how the **airflow** is driven (in open doors and room with stove)
  3. Prove the importance of **open doorways** and **construction mode**
  4. **Measure thermal environment** in a same passive house with different stoves
  5. **Develop and validate a simple modeling procedure** to evaluate the all-year thermal comfort at an acceptable computational cost
  6. **Simulate integration of several different stoves** within a typical passive house (in Norway and Belgium)
- What remains to be done:
  - Capture the **vertical stratification** in simulation (e.g. using CFD)
  - Model and investigate the influence of the stove **glazing**
  - A **quick-prototyping tool** is close to be ready
  - Influence of the addition of **phase-change material (PCM)** in the stove envelope



The Research Centre on  
Zero Emission Buildings

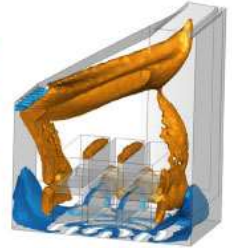


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## WoodCFD



Clean and efficient wood stoves through improved batch combustion models and CFD modelling approaches

# Thank you for your attention!

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