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3D CFD modelling of solid biomass combustion in grate furnaces

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Scope of work

- Development of a **CFD based model for packed bed combustion** in biomass grate furnaces
- A model for the thermal conversion of the packed bed should include models for
 - the particle movements in the fuel bed
 - the thermal conversion of the particles
- Link of the packed bed model and link with an in-house developed CFD model for the turbulent reactive flow in the combustion chamber
- Application and test of the model with a 20 kW underfeed stoker grate furnace and a pilot-scale moving grate furnace (180 kW)

Methodology



Modelling of the thermal conversion of packed beds can be divided into two parts:

1. Thermal conversion of particles

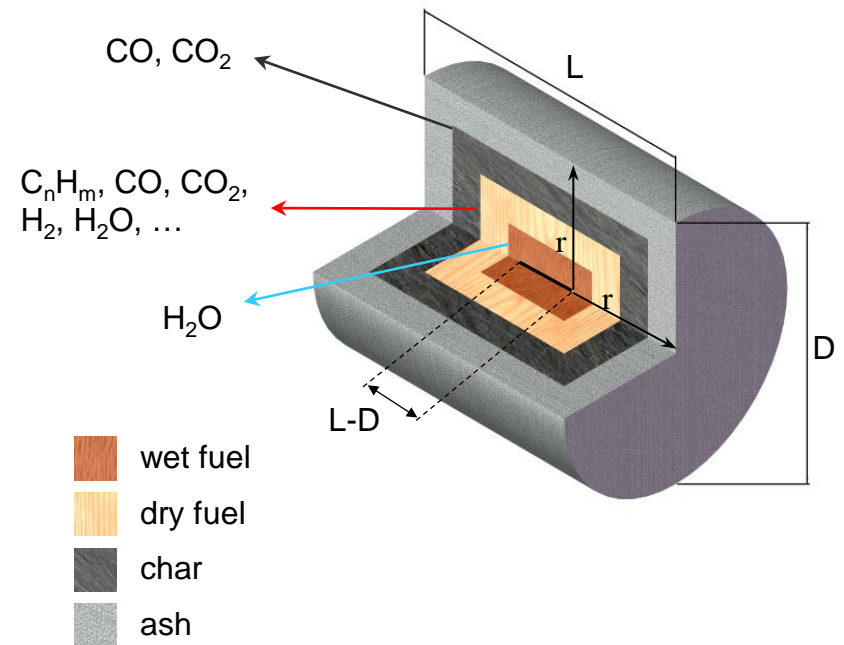
- A **layer model**, implemented as UDF in ANSYS FLUENT, has been developed to consider the biomass particles as **thermally thick** particles with intra-particle gradients.

2. Hydrodynamics of packed beds

- The **Euler-Granular** model of ANSYS FLUENT is used, since it allows to consider **particle-particle collisions**.
- The Euler-Granular model does not enable to hook the layer model.
- The velocity field of the particle phase from the Euler-Granular **simulation without chemical reactions** is saved and is then used in the subsequent **simulation of particle combustion with the Discrete Phase Model**.

Thermal conversion of particles (layer model) (I)

- Spherical and cylindrical particles can be modelled by the layer model
- A particle is discretised in four layers
- Each layer corresponds to a conversion stage (wet fuel, dry fuel, char, ash)
- Boundaries between the layers correspond to the conversion fronts of the sub-processes (drying, pyrolysis, char burnout), which move towards the particle centre during conversion
- Simultaneous progress of the sub-processes is possible
- **Advantage:** the consideration of the intra-particle gradients improves the results of particle conversion rates
- The model is published in Fuel Processing Technology, 95 (2012), pp. 96-108.

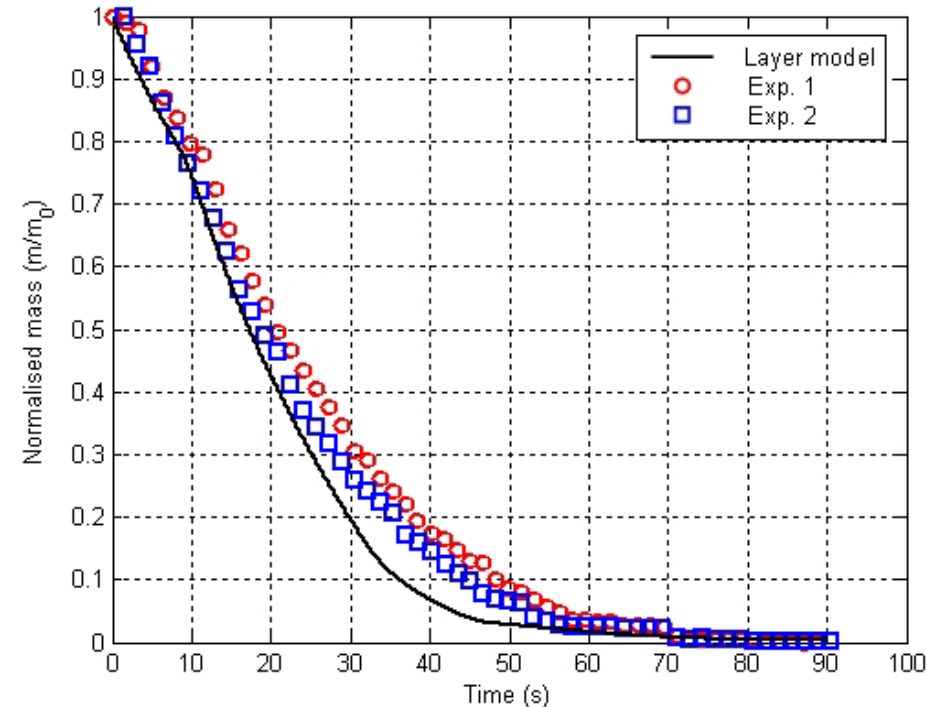
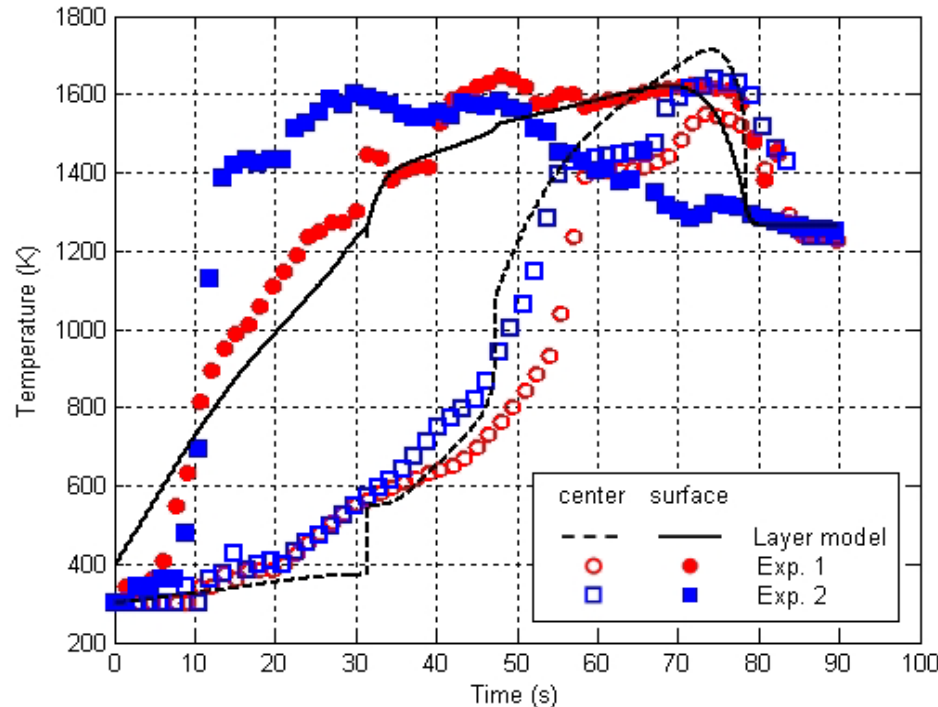




Thermal conversion of particles (layer model) (II)

- Drying is assumed to take place at a constant temperature (the boiling temperature of water). Therefore, the drying rate is limited by the heat transfer.
- Pyrolysis is described by three independent competitive reactions for cellulose, hemicellulose, and lignin.
 - The kinetic rate of each reaction is calculated by an Arrhenius equation.
 - The empirical constants are obtained from fuel specific fast heating rate TGA experiments.
- Char oxidation and gasification reactions are assumed to be limited by the mass transport rate.
 - CO and CO₂ are the products of char oxidation.
 - Char gasification with H₂O, CO₂ and H₂ is also considered.

Validation of the layer model



Experimental setup:

- Single particle reactor, oxidising conditions
- Fuel: poplar wood (cylindrical particle)
- Water content: 40 (wt% w.b.)
- Diameter: 9.5 (mm); length = 9.5 (mm)

- Reactor wall temperature: 1276 (K)
- Air temperature: 1050 (K)

Exp. data: Lu H., *Experimental and modeling investigation of biomass particle combustion*, Ph.D. thesis, 2006, Brigham Young University.

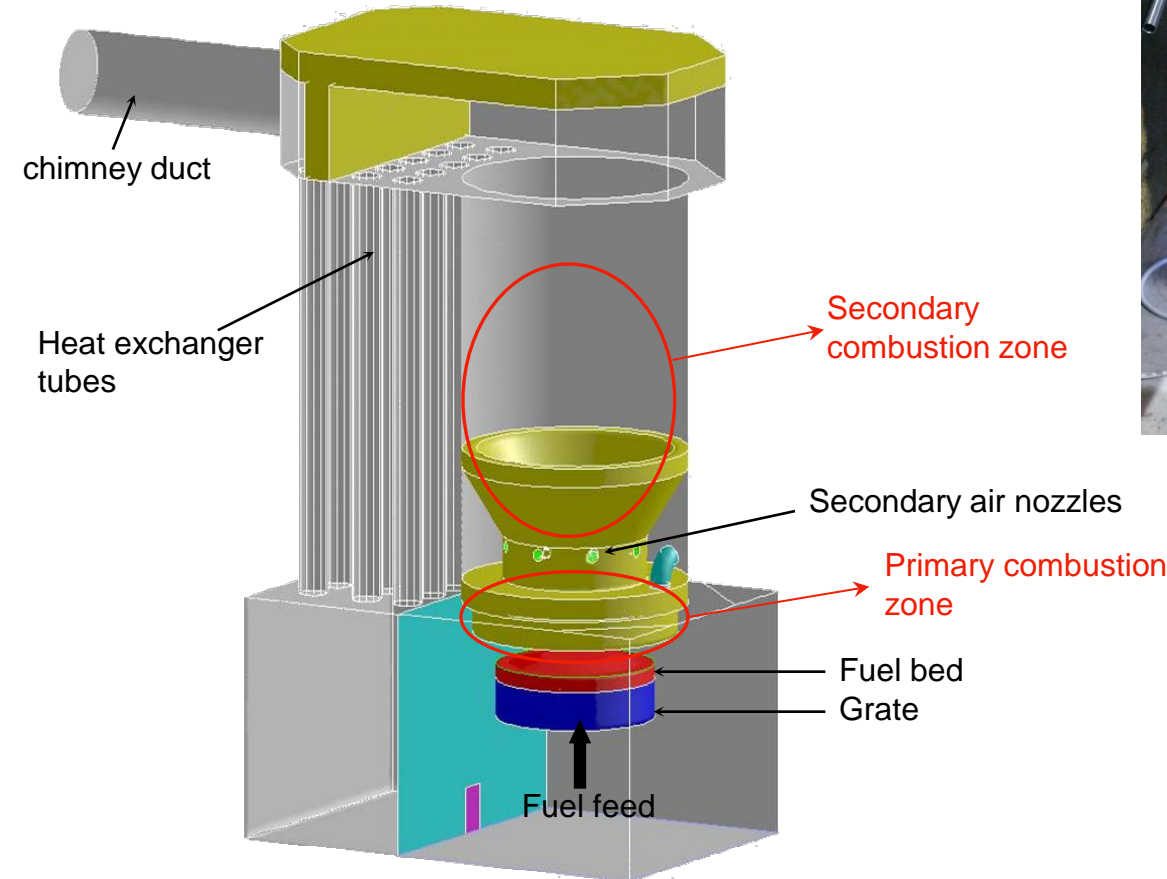
Hydrodynamics of the packed bed

- The Euler-Granular model assumes that the particle phase behaves as a fluid. Therefore, the **continuity** and **momentum** equations are solved for both gas and particle phases:
- An additional conservation equation is formulated for the **granular temperature**:

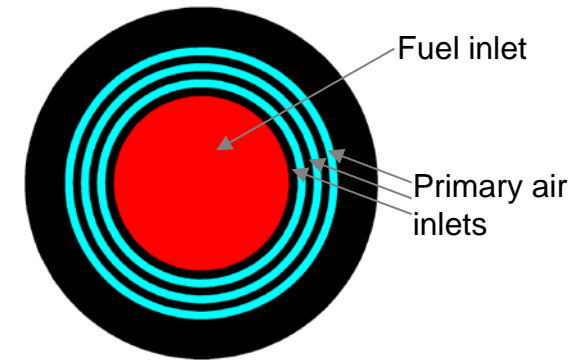
$$\frac{3}{2} \left[\frac{\partial}{\partial t} (\alpha_s \rho_s T_G) + \nabla \cdot (\alpha_s \rho_s \vec{u}_s T_G) \right] = \underbrace{(\bar{\tau}_s - p_s \bar{I}) : \nabla \vec{u}_s}_{\text{Production}} + \underbrace{\nabla \cdot (\kappa \nabla T_G)}_{\text{Diffusion}} - \underbrace{\gamma}_{\text{Dissipation}} - \underbrace{3K_{gs} T_G}_{\text{Exchange}}$$

- The granular temperature T_G is associated with the **kinetic energy** of the **particle fluctuation** around the mean velocity.
- To close the governing equations the **constitutive equations** are required for the following parameters: solid stress tensor $\bar{\tau}_s$, solid pressure p_s , diffusion coefficient κ , collisional dissipation γ and momentum exchange coefficient K_{gs} .

Example 1: simulation of 20 kW underfeed stoker furnace



Picture of the grate



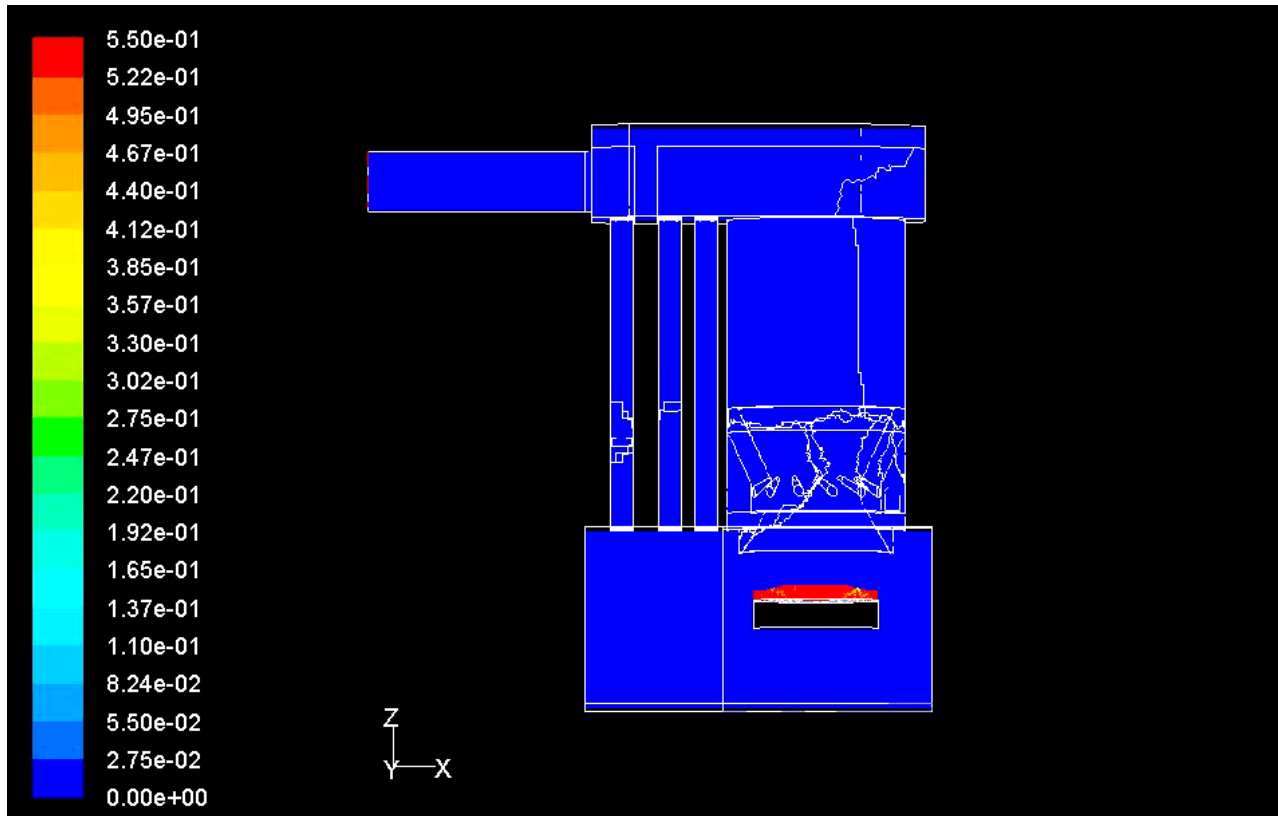
Scheme of the grate used for simulations

Operating data: Biomass fuel: softwood pellets;

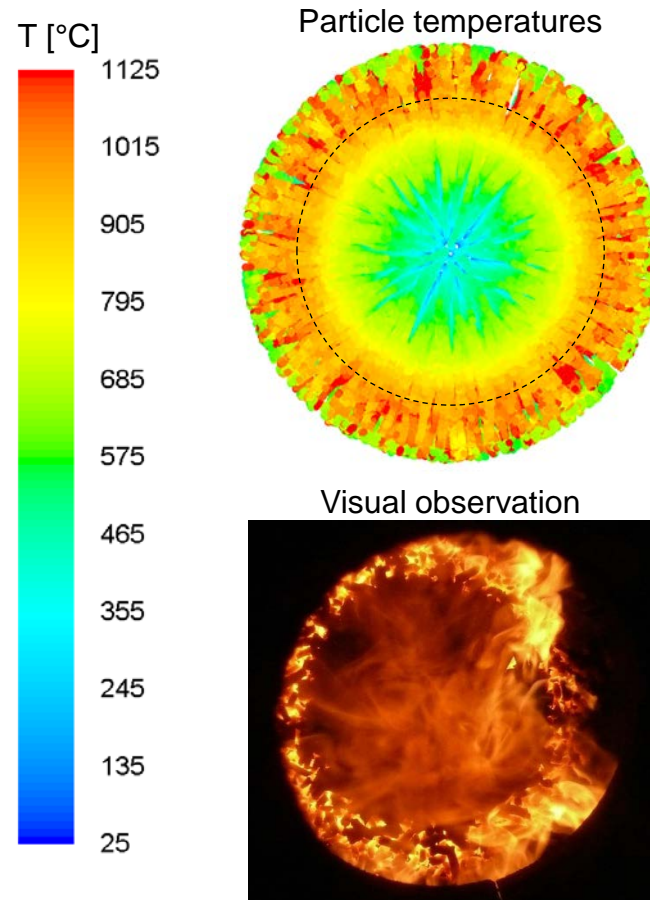
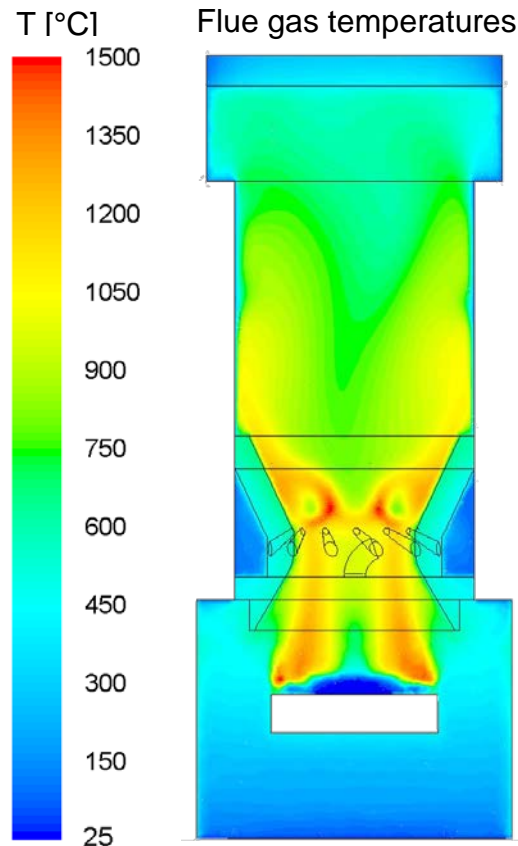
Moisture content= 8.1 wt.% (wet base); $\lambda_{\text{total}} = 1.58$; $\lambda_{\text{prim}} = 0.64$, no flue gas recirculation



Particle motion on the grate

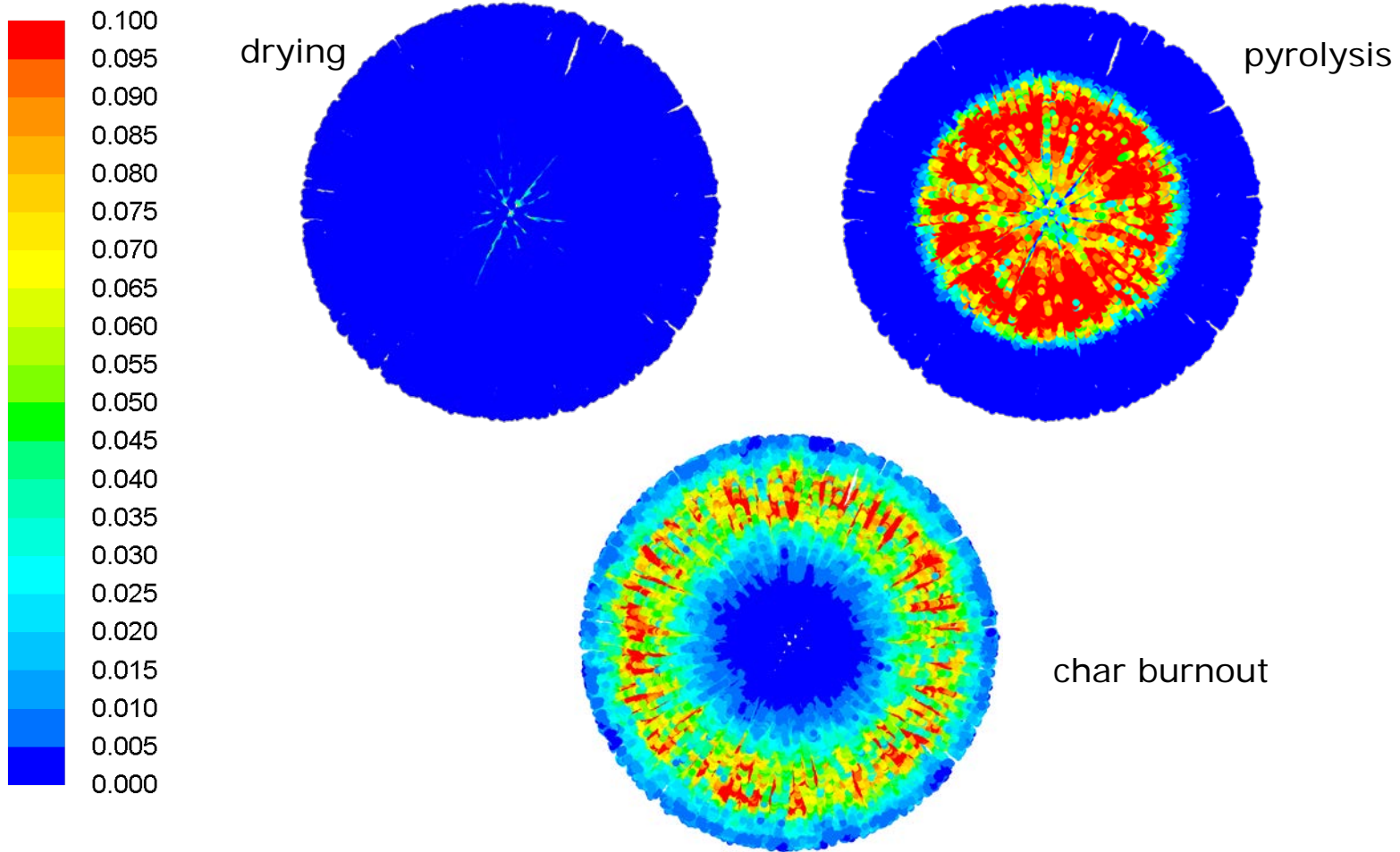


Particle and flue gas temperatures [°C]



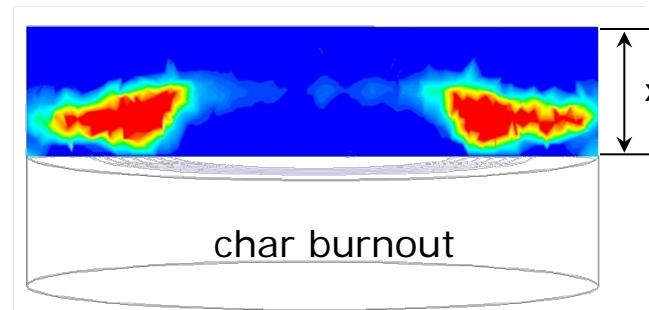
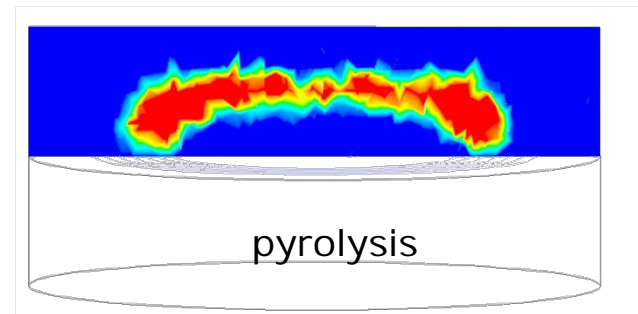
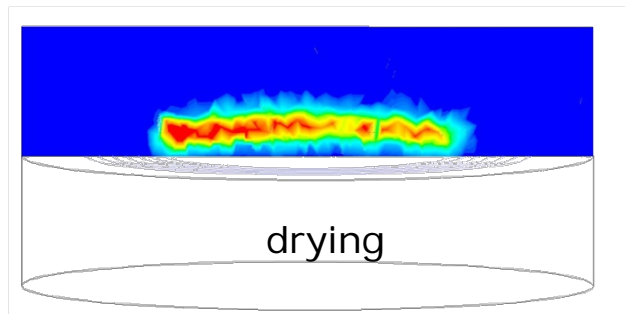
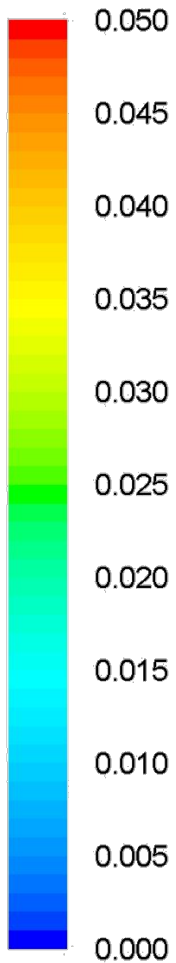
Explanations: Biomass fuel: softwood pellets;
Moisture content= 8.1 wt.% (wet base); $\lambda_{\text{total}} = 1.58$; $\lambda_{\text{prim}} = 0.64$, no flue gas recirculation

Particle tracks coloured by release rates [mg/s]



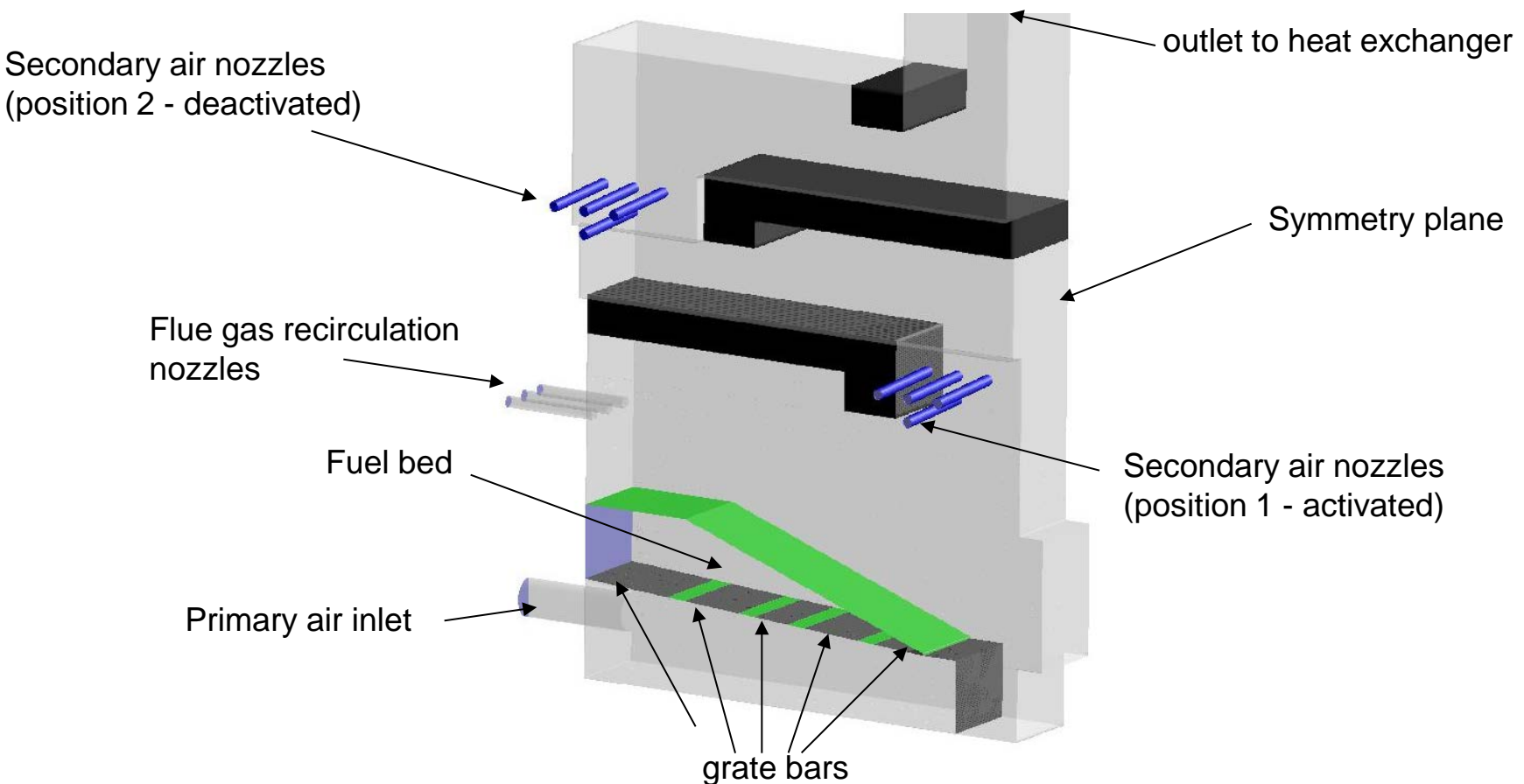
The concentration of unburned char in the ash is less than 1 wt.% (d.b.), which is in good agreement with experimental results.

Contours of release rates [mg/s]



Contours of release rates at a vertical cross section through the grate axis; $x = 5\text{cm}$

Example 2: simulation of a 180 kW moving grate furnace

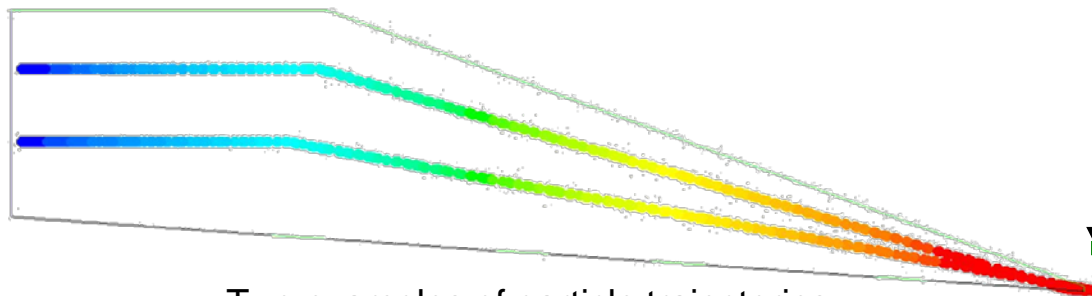


Operating data: Biomass fuel: spruce wood chips;
Moisture content= 21.8 wt.% (wet base); $\lambda_{\text{grate}} = 0.69$; $\lambda_{\text{PCC incl. FGR}} = 0.9$; $\lambda_{\text{total}} = 1.67$

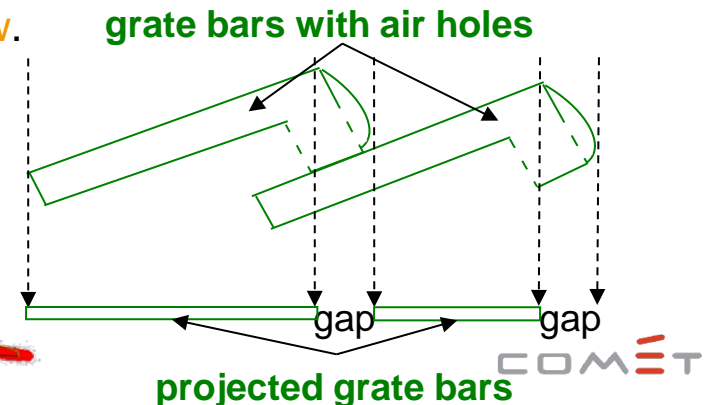
Methodology

Grate position & simplification

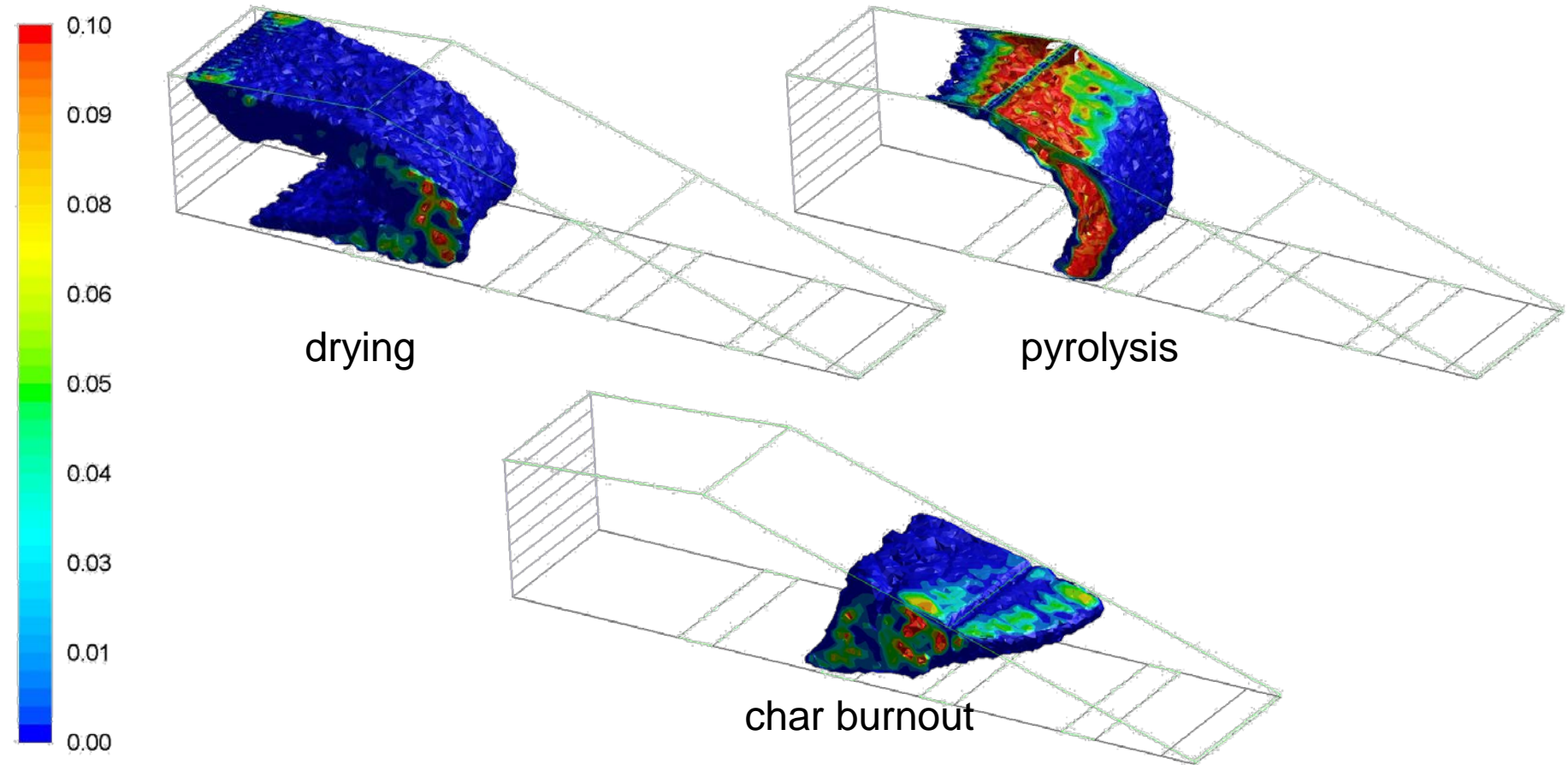
- Even in steady state operations, a **time-dependent variation** of fuel conversion is taking place due to the grate movements. For the selected test run, a grate position has been chosen, where **the conversion was at its average value**.
- After the position of the grate was chosen, its geometry was simplified by projecting it to a plane, as it is shown below.
- Due to the simplification of grate bars, **the particle velocities are pre-defined** and together with their trajectories are included in a UDF and coupled to ANSYS Fluent. Two particle paths are shown below.
- Primary air is introduced below the grate, where it is distributed and passes through the gaps in the grate. During the flow through the gaps, **the air is heated by the grate bars**.
- **The pressure drop** through the bed was also modelled by treating it as a porous medium. The coefficients were defined according to **Ergun's law**.



Two examples of particle trajectories

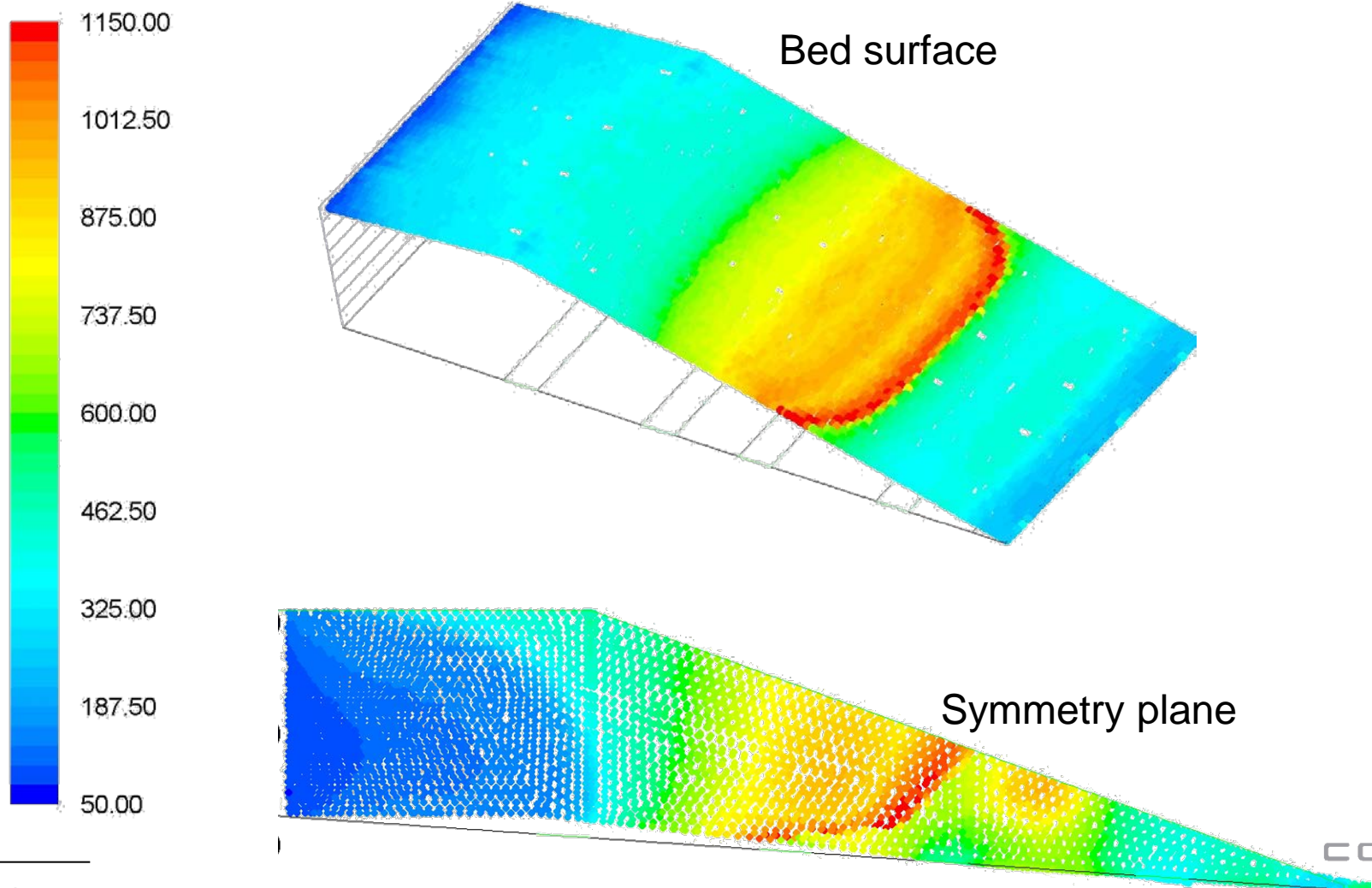


Release zones inside the fuel bed coloured by the release rates [mg/s]



Half of the grate width is shown; from the wall to the symmetry plane.

Particle temperature at the surface of the fuel bed and in the symmetry plane [°C]

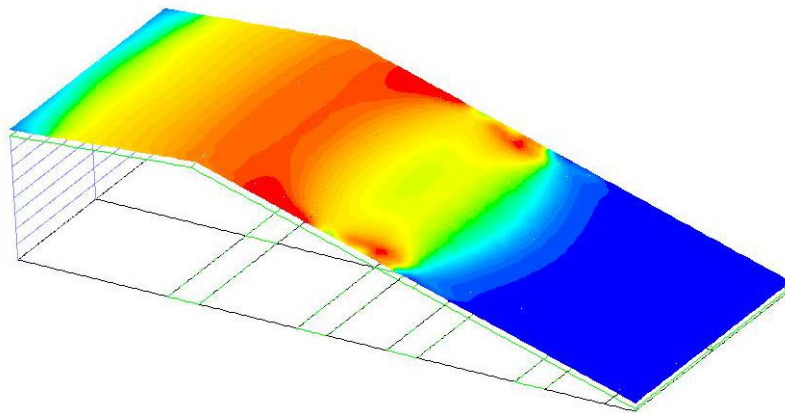


CO and CH₄ (vol.% d.b.) concentrations at the surface of the fuel bed



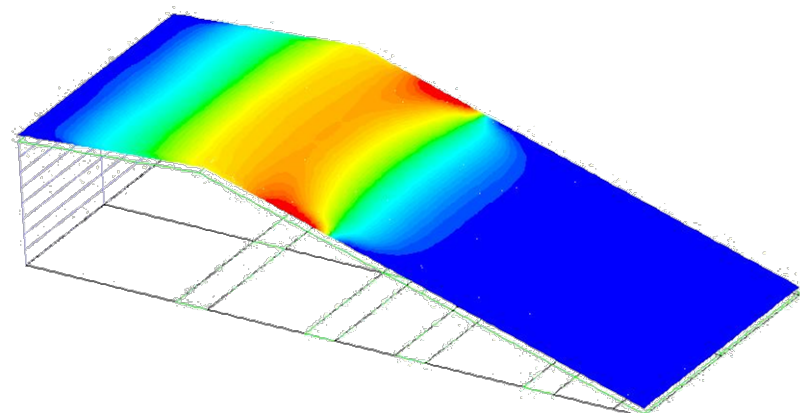
CO

40.00
35.00
30.00
25.00
20.00
15.00
10.00
5.00
0.00



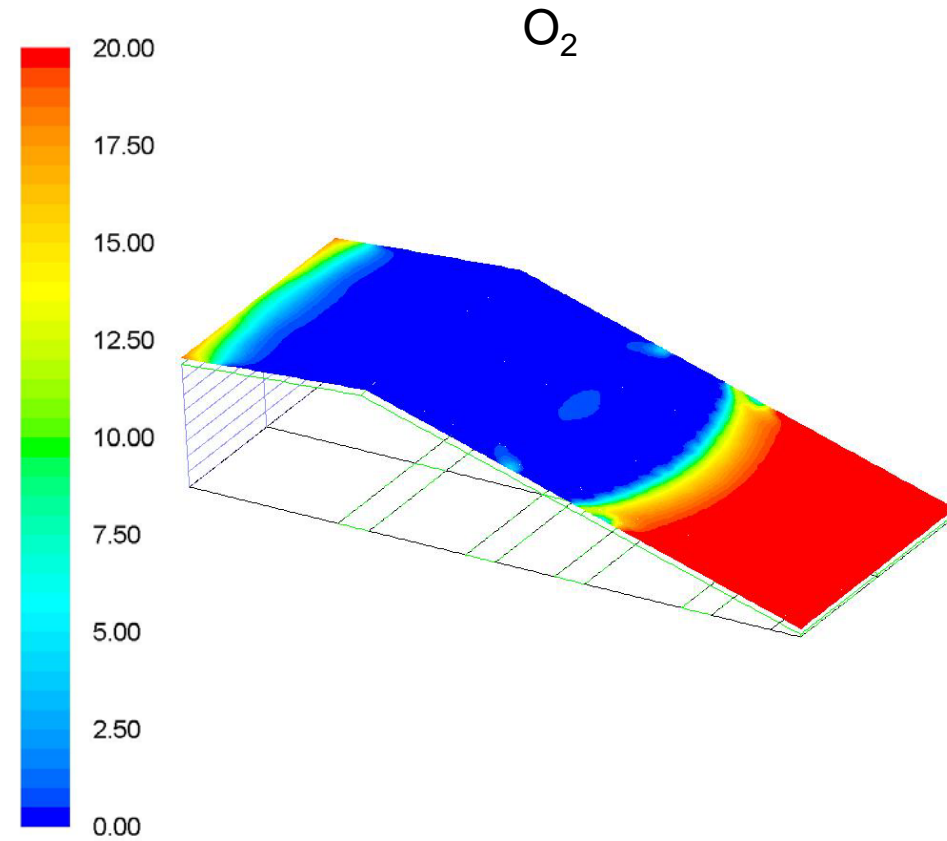
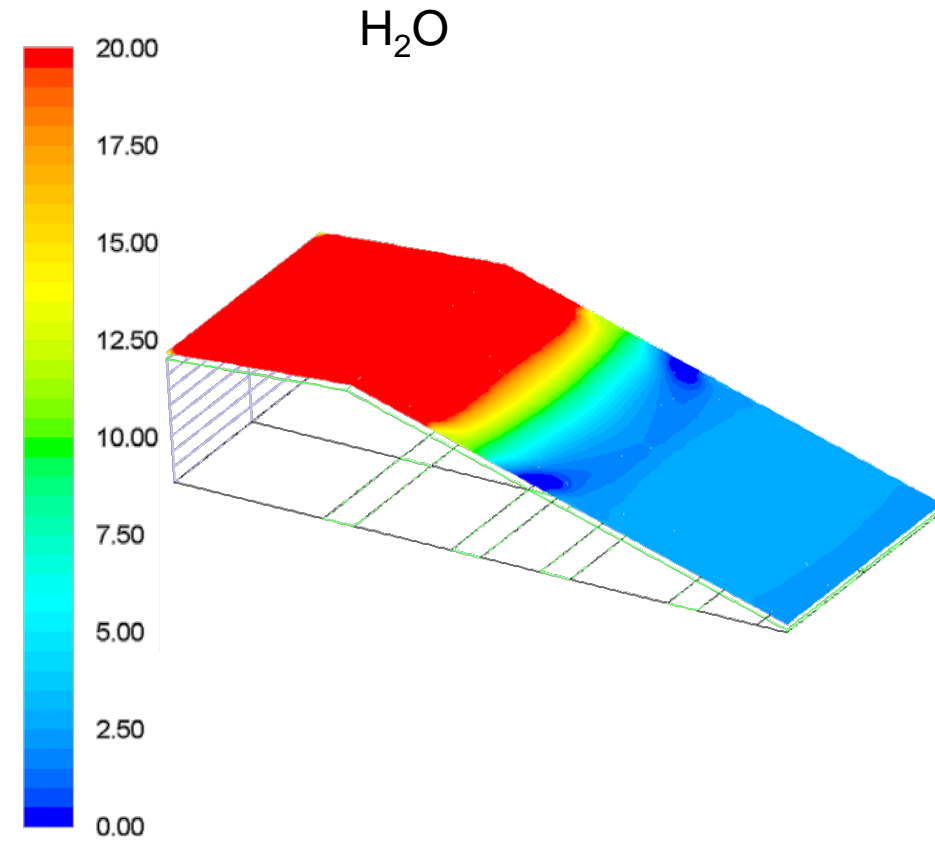
CH₄

30.00
26.25
22.50
18.75
15.00
11.25
7.50
3.75
0.00





H₂O (vol.% w.b.) and O₂ (vol.% d.b.) concentrations at the surface of the fuel bed





Summary and conclusions

A CFD based model for packed bed combustion in biomass grate furnaces has been developed and successfully tested. The model has the following advantages:

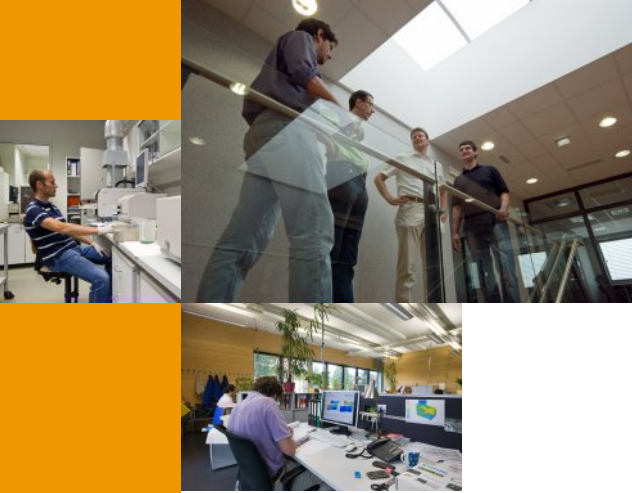
- Approximating the particle movements on the grate under consideration of **particle-particle collisions** (already applied for the fixed grate and is on-going for the moving grate).
- Due to the consideration that the particles are **thermally thick**, the particle temperatures and consequently the mass loss rate during the thermal conversion of the particles can be better predicted.
- The effect of **particle related parameters** (size, physical properties and moisture content) as well as **operating conditions** (air distribution below the grate, air pre-heating, flue gas re-circulation and air staging) can be investigated.
- The packed bed modelling has been **linked with an in-house developed gas phase CFD model for turbulent reactive flows in the combustion chamber**. Hence, also the influence of flow, gas phase combustion and heat transfer from the combustion chamber on packed bed conversion is considered.



Outlook

Further developments planned:

- Development of an **enhanced heat transfer** model for the packed bed (considering particle-particle radiation and conduction)
- Application of an appropriate gas-solid multiphase model to **simultaneously simulate** the movement and the thermal conversion of the particles.



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Thank you for your attention

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